

NELINEARNA ANALIZA STABILNOSTI OKVIRNIH NOSAČA

NONLINEAR STABILITY ANALYSIS OF THE FRAME STRUCTURES

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1 UVOD

Problemi gubitka stabilnosti armiranobetonskih, a još više čeličnih konstrukcija, veoma su aktuelni, a posebno imajući u vidu želje projektnata da grade atraktivne objekte velikih visina i raspona, odnosno velike vitkosti. Proračun ovakvih objekata, posebno iz aspekta analize njihove stabilnosti, zahteva primenu složenih numeričkih modela. Iako postoji značajan broj radova u literaturi, posvećenih raznim problemima stabilnosti konstrukcija, i dalje ima dosta nerešenih ili nedovoljno rešenih problema, posebno kada je reč o ponašanju realnih građevinskih konstrukcija u elasto-plastičnoj oblasti.

Istraživanja u oblasti stabilnosti linijskih nosača, počevši od prvih radova Euler-a krajem osamnaestog veka pa sve donedavno, uglavnom su se bazirala na rešavanju diferencijalnih jednačina izvijanja štapa izvedenih prema teoriji drugog reda. Da bi se taj problem lakše rešio kada su u pitanju složene konstrukcije, istraživači su vršili određena uprošćenja tako da su, na primer, konstrukcije podelili na sisteme štapova s nepomerljivim čvorovima i sisteme štapova s pomerljivim čvorovima. Takođe su posebno izučavali štapove sa elastičnim uklještenjima na jednom ili oba kraja, i to u kombinaciji sa elastičnim osloncima ili bez njih. Numerički izrazi i grafički prikaz dobijenih rezultata za kritično opterećenje dati su, na primer, u [1], [12]. Korišćenjem navedenih izraza, na relativno jednostavan način može da se obavi i proračun višespratnih okvira. Metode koje se najčešće koriste u tom slučaju jesu proračun korišćenjem jednačina obrtanja i korišćenjem izraza za rotacionu krutost [1], [17]. Navedeni postupak

1 INTRODUCTION

The problems of instability of reinforced concrete structures and, even more, of steel structures, are very contemporary, particularly having in mind desires of engineers to build attractive tall structures with high slenderness. Design of these structures, especially from the viewpoint of their stability, requires an application of complex numerical models. Although there are a significant number of papers in the literature devoted to the various problems of structural stability, there are still a lot of unsolved or inadequately solved problems, especially in the case of the real behavior of structures in elasto-plastic domain.

Research of the stability of linear structures, starting from the first Euler's investigations at the end of the eighteenth century, until recently, was mainly based on solving the differential equation of buckling according to the second order theory. In order to find simple solutions for the more complex structures, the researchers performed some approximations in their calculation. It means that members which are "isolated" from the structure, with different boundary conditions, were analyzed. Also, separately sway and non-sway structures were considered. Numerical expressions and graphical representation of the obtained results for the critical load can be found, for example in [1], [12]. Such approximations were also used to formulate procedures for calculations of multi-story frames. The most used methods for this type of calculation are slope deflection method and stiffness distribution method, and they can be found in the literature, for example [1], [17]. The above procedures

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proračuna višespratnog okvira s jednim poljem može da se primeni i kada su u pitanju znatno složenije konstrukcije okvira s više polja. Naime, u ovom slučaju se problem okvira s više polja može svesti na proračun ekvivalentnog okvira s jednim poljem, kao što je prikazano u [22], [23]. U opštem slučaju, zbog različitih dimenzija stubova i rigli, kao i različitog opterećenja, višespratni okviri s više polja ne mogu da se zamene jednim ekvivalentnim okvirom i da se na taj način odredi njihovo kritično opterećenje. Zato su pojedini autori rešenje problema potražili u primeni nekih drugih metoda, kao npr. energetskog postupka [16].

Teorijske osnove proračuna izolovanog štapa bile su baza za donošenje nacionalnih i evropskih propisa o stabilnosti okvirnih nosača [10], [11] i [25]. Međutim, primena ovih propisa ukazala je na to da takav proračun u pojedinim slučajevima dovodi do grešaka, jer je on približan [7]. Usled toga, poslednjih godina radi se na poboljšanju ovih približnih metoda proračuna. Tako se, na primer, navodi analiza [15] koja ima cilj da se poboljšaju ulazni parametri koji definišu krutost okvirnog nosača, a samim tim i tačnije odrede koeficijenti efektivne dužine izvijanja kod višespratnih okvira.

Na kraju ovog uvoda, treba naglasiti primenu metode konačnih elemenata kao najefikasnije metode za numeričku analizu stabilnosti okvirnih nosača. Naime, kao što je poznato, u linearnoj teoriji prvog reda matrica krutosti konačnog elementa zavisi od geometrije elementa i od mehaničkih karakteristika materijala. Kad je u pitanju problem stabilnosti, on ne može da se reši prema teoriji prvog reda i potrebno je sprovesti proračun prema teoriji drugog reda. To ima za posledicu da je neophodno u proračun uvesti matricu krutosti koja zavisi i od aksijalnih sila. Uobičajeno uprošćenje u ovom proračunu prema teoriji drugog reda jeste da se problem aksijalnog naprezanja i problem savijanja štapa definišu nezavisno jedan od drugog. To omogućuje da se i određivanje matrice aksijalne i transverzalne krutosti štapa mogu razmatrati razdvojeno, kao dva međusobno nezavisna problema. Primenom metode konačnih elemenata u analizi stabilnosti linijskih nosača bavili su se mnogi istraživači, kao npr. [14], [2], a takođe je primenjena u savremenim komercijalnim programima za ovu vrstu analize (SAP2000, STAAD...). Treba napomenuti da se standardno rešenje metode konačnih elemenata dobija preko geometrijske matrice krutosti. U ovom radu biće dato rešenje gde su matrice krutosti izvedene korišćenjem interpolacionih funkcija koje se odnose na tačno rešenje diferencijalne jednačine savijanja štapa prema teoriji drugog reda. Takođe, pri proračunu se osim geometrijske uvodi i materijalna (fizička) nelinearnost, pa su matrice krutosti izvedene korišćenjem tangentnog modula elastičnosti koji prati promenu krutosti štapa u neelastičnoj oblasti.

Pristup koji se zasniva na primeni teorije tangentnog modula u poslednje vreme dosta je razmatran u literaturi, i pri tome može da se izdvoji rešenje koje je dato u [24]. Rezultati dobijeni u ovom radu pokazuju da se predloženom analizom stabilnosti u neelastičnoj oblasti može izračunati kritična sila čeličnih okvirnih nosača i na taj način obaviti proračun takvih nosača.

for calculation of one-bay multi-storey frames can be applied for more complex multi-bay frames. Namely, in this case, the calculation of multi-bay framework can be reduced to calculation of equivalent one-bay frame, as it is shown in [22], [23]. Sometimes, the framework because of its irregularity can not be reduced to an equivalent single-bay frame in order to calculate their critical load. In that case, some other procedure, based on the linear elastic analysis, such as energy method [16], are suggested.

Theoretical approach based on the calculation of isolated member was applied in the national and European regulation for the stability of frame structures [10], [11] and [25]. However, the application of these codes shows that such calculation, in some cases, may lead to the substantial errors, because obtained results are approximate [7]. Therefore, in recent years, considerable effort has been made in order to improve these approximate calculation methods. So, for example, the objective of the analysis [15] is to propose improved input parameters for the determination of the effective buckling length coefficient of columns in multi-story frames.

At the end of this introduction, it should be emphasized application of the finite element method as the most effective method for numerical analysis of stability of frame structures. Namely, it is well known that in the linear first order theory, stiffness matrix is the function of geometry of the element and the characteristics of the material. However, stability problem can not be solved by the first order theory, and it is necessary to perform calculation according to the second order theory. Therefore, calculation should be performed using the stiffness matrix which depends upon the axial forces in the element. Usual simplification in this calculation according to the second order theory is that the axial loading and bending problems are considered independently from each other. This enables that the determination of the matrix of axial and transverse stiffness of the member can be considered as two separate problems. The finite element method was investigated by many authors, for example [14], [2], and also it is used in modern commercial programs for such kind of analysis (SAP2000, STAAD, ...). In the usual approach, the finite element method is based on the geometric stiffness matrix as a part of the tangent stiffness matrix. In this paper it is given solution where stiffness matrix is derived using interpolation functions related to the exact solution of the differential equation of bending of a beam according to the second-order theory. Also, in this analysis, in addition to geometric nonlinearity, the materially (or physically) nonlinear analysis is also taken into account. It means that stiffness matrices are derived using the tangent modulus that is stress dependent and follows changes of the member stiffness in the inelastic domain.

The approach that is based on the application of tangent modulus theory is also discussed in the literature, and solution given in [24] can be specified. The results obtained in this paper show that the proposed inelastic buckling analysis suitably evaluates the critical load and failure modes of steel frames, and can be a good alternative for the evaluation of critical load in the design of steel frames.

2 STABILNOST OKVIRNIH NOSAČA U ELASTO-PLASTIČNOJ OBLASTI

Kao što je poznato, pri izvijanju štapa dolazi do njegovog savijanja usled aksijalne sile. Pri rešavanju problema stabilnosti, diferencijalna jednačina ravnoteže štapa koristi se u obliku:

$$v^{iv} + k^2 v'' = 0 \quad (1)$$

gde je $k = \sqrt{P/EI}$, P je aksijalna sila, EI krutost štapa na savijanje, dok v predstavlja ugib normalan na prvobitnu (nedefinisano) osu štapa. Kada se izvijanje dešava u elastičnoj oblasti, modul elastičnosti E ima konstantnu vrednost.

Kao što je već istaknuto, u metodi konačnih elemenata uobičajeno je da se ova diferencijalna jednačina rešava koristeći približno rešenje u obliku polinoma. Međutim, cilj ovog istraživanja je da se dođe do tačnih rešenja problema stabilnosti okvirnih nosača, tako da se ovde koristi tačno rešenje diferencijalne jednačine (1), koje je dato preko trigonometrijskih funkcija:

$$v(x) = \alpha_1 + \alpha_2 kx + \alpha_3 \sin(kx) + \alpha_4 \cos(kx) \quad (2)$$

Matrica krutosti koja se dobija po teoriji drugog reda za štap koji je opterećen silom pritiska, odnosno silom zatezanja prikazana je, na primer, u [6].

Numerički primjeri u kojima je analizirana primena dobijene matrice krutosti dati su u [7], [6]. Pri tome je kritično opterećenje dobijeno kao koren transcendentne jednačine, koja predstavlja uslov da je determinanta odgovarajuće matrice krutosti jednaka nuli. Iz dobijenih rezultata zaključeno je da se velike greške mogu javiti kada se primenjuje klasična (linearna) metoda konačnih elemenata. Treba napomenuti da ove greške mogu da se smanje ako se broj konačnih elemenata dovoljno poveća. Međutim, problem ovog približnog postupka je u tome što je potrebno stalno vršiti kontrolu da bi se videlo koliki broj konačnih elemenata je potreban da bi se dobilo konvergentno rešenje. Ovo je i razlog zašto su u ovoj analizi korišćene interpolacione funkcije u trigonometrijskom ili hiperboličkom obliku. Glavna prednost takvog pristupa je u tome što daje, uslovno rečeno, tačna rešenja i ukupan broj konačnih elemenata je pet do deset puta manji nego u uobičajenom postupku s primenom geometrijske matrice krutosti. Nedostatak je u tome što umesto problema sopstvenih vrednosti, za čije rešavanje postoje nekoliko dobro poznatih postupaka (npr. iteracije unutar prostora, Lanczos-ov postupak, itd...) problem izvijanja svodi se na rešavanje transcendentne jednačine koja je funkcija, na veoma komplikovan način, aksijalne sile u stubovima i gredama. Zato je i formulacija odgovarajućih algoritama i kompjuterskog programa za rešavanje ovakve vrste problema jedan od glavnih naučnih doprinosa ovog istraživanja.

Kao što je već rečeno, analiza problema stabilnosti zasniva se na proračunu prema teoriji drugog reda. To znači da se razmatra geometrijski nelinearan problem zato što su uslovi ravnoteže napisani na deformisanoj konfiguraciji nosača, odnosno uzima se u obzir izmjenjena geometrija nosača do koje se dolazi usled deformacija nastalih pod zadatim opterećenjem. Kada je u pitanju veličina naprezanja u pojedinim štapovima nosača, pretpostavljeno je da je posredi problem elastične stabilnosti, odnosno da sve vreme do dostizanja

2 STABILITY OF THE FRAMES IN ELASTO-PLASTIC DOMAIN

It is well known that during the member buckling, axial force produces the bending of the member. The basic differential equation of this stability problem is:

where k is equal to $\sqrt{P/EI}$, P is axial force, EI is member bending stiffness, and v represents lateral deflection. When the buckling occurs within the elastic range, the modulus of elasticity E has a constant value.

As it is well known, in the finite element method it is usual to use approximate polynomial solution for this differential equation. However, the main aim of this investigation is to apply exact solutions on the problem of stability of frame structures. In order to formulate that exact matrix stability analysis, shape functions are used in the trigonometric form, according to the solution of equation (1):

Obtained stiffness matrix for the members subjected to compressive and tension forces can be found, for example, in [6].

Numerical analysis related to this problem is given in [7], [6]. The critical buckling loads are obtained from the roots of the transcendental equation, representing the condition that the determinant of the corresponding stiffness matrix is equal to zero. From the results of performed analysis it was concluded that when the classic (linear) finite element method is used, large errors for values of critical load might be obtained. It should be noted that these errors could be reduced, if the number of member elements in approximate solution is increased. But, the problem of this approximate procedure is also that it is necessary to perform previous control analyses in order to obtain how many finite elements are needed for convergent solution. These arguments present the reason why in this analysis the interpolation polynomials in the form of trigonometric or hyperbolic functions, sometimes known as the stability functions, are used. The main advantage of such an approach is that it gives, conditionally speaking, exact solutions and the total number of finite elements is 5-10 times less than in the usual approach based on the geometric stiffness matrix. Disadvantage is obvious: instead of the generalized eigenvalue problem, for which there are several well established methods (e.g. the subspace iteration, Lanczos method, etc), the buckling problem is reduced to the solution of the transcendental equation which depends, in a very complicated way, upon the normal forces in columns and beams. So, formulating the suitable algorithms and corresponding computer program for solving such kind of problems is one of the main scientific contributions of this analysis.

As it was already mentioned, the stability analysis is performed according to the second order theory. It means that geometrically nonlinear problem is analyzed since the equilibrium conditions are applied on the deformed configuration of the member. Taking into consideration the stress value in the analyzed columns, it is assumed that this is a problem of elastic stability. It

kritične sile naponi (σ_{cr}) u svim štapovima ne prelaze granicu proporcionalnosti (σ_p) materijala od koga su napravljeni.

Proračun na bazi teorije elastične stabilnosti široko je primjenjen u inženjerskoj praksi, pošto se polazi od toga da se građevinske konstrukcije uglavnom ponašaju elastično kada su izložene svakodnevnim eksplatacionim opterećenjima. Zato je razumljivo da ovaj vid proračuna predstavlja osnovu standarda (propisa) za analizu stabilnosti okvirnih konstrukcija [10], [11] i [25]. On je definisan kroz određivanje tzv. efektivne dužine izvijanja pojedinih šapova okvirnih nosača.

Proračun stabilnosti okvirnih nosača komplikuje se ukoliko pre dostizanja kritičnog opterećenja pojedini štapovi uđu u fazu nelinearnog ponašanja materijala. To znači da se u njima javljaju naponi koji su veći od granice proporcionalnosti. Time ovaj proračun dobija još jedan vid nelinearnosti zato što postaje i materijalno (fizički) nelinearan problem.

Kao što je poznato, polazeći od Euler-ove kritične sile, kritični napon u štalu može se predstaviti u funkciji modula elastičnosti (E) i vitkosti (λ_i):

$$\sigma_{cr} = \frac{P_{cr}}{A} = \frac{\pi^2 EI}{l_i^2 A} = \pi^2 \frac{E}{\lambda_i^2} \quad (3)$$

gde su: A - površina poprečnog preseka, $i = \sqrt{I/A}$ - poluprečnik elipse inercije i λ_i - vitkost štapa.

Jednačina ove hiperbole važi sve dok je kritični napon manji od granice proporcionalnosti, slika 1. Kada je ovaj napon prekoračen, izvijanje se dešava u plastičnoj oblasti. Proučavanjem stabilnosti štapova koji se izvijaju u plastičnoj oblasti prvi se bavio Bauschinger koji je vršio eksperimentalna istraživanja krajem devetnaestog veka. Na bazi ovih rezultata i sopstvenih istraživanja, Tetmajer je nešto kasnije prvi dao izraz za vezu napona i vitkosti u plastičnoj oblasti. On je predložio linearnu vezu u obliku:

$$\sigma_{cr} = \sigma_v \quad \text{za } 0 < \lambda < 60$$

$$\sigma_{cr} = C_1 - C_2 \cdot \lambda \quad \text{za } 60 < \lambda < \lambda_p$$

gde je $\lambda_p = \pi \sqrt{E/\sigma_p}$ vitkost na granici proporcionalnosti.

Grafički prikaz ove veze dat je na slici 1.

U okviru daljih istraživanja u ovoj oblasti ističe se i rad Engesser-a [9] koji uvodi pojam tangentnog modula elastičnosti. Kasnije su se i mnogi drugi naučnici bavili ovom problematikom. Tako su na primer Karman i Shanley izvršili modifikaciju Engesser-ove krive. Neke od najznačajnijih krivih izvijanja u plastičnoj oblasti prikazane su na slici 2.

I pored brojnih istraživanja, kako eksperimentalnih tako i teorijskih, problem izvijanja štapova, a posebno okvirnih nosača u elasto-plastičnoj oblasti nije do sada u potpunosti rešen. Tek se s razvojem kompjuterske tehnike stvorila mogućnost za svedobuhvatno rešenje ovog problema. Treba napomenuti da su se ovim problemom u poslednje vreme bavili mnogi autori čiji su rezultati prikazani u [24], [5] i [13].

means that when the critical load is reached, stresses (σ_{cr}) in all columns do not exceed the proportionality limit of the material (σ_p).

Calculation based on the elastic stability theory is widely applied in the engineering practice, because it can be assumed that engineering structures have generally elastic behavior when they are subjected to the usual working loads. Therefore, it is clear that such theory is the basis of the standards for the stability analysis of the frame structures [10], [11] and [25], and this calculation is defined by the determination of the effective buckling length of the compressed columns.

Stability calculation becomes more complicated if, before the critical load is achieved, some compressed members enter into the phase of nonlinear material behavior. It means that stresses in such columns become higher than the proportionality limit. Therefore, such calculation obtains another type of nonlinearity and it becomes also materially (or physically) nonlinear problem.

Taking into consideration well known expression for the Euler's critical force, critical stress in a member may be expressed as a function of the modulus of elasticity (E) and the slenderness ratio (λ_i):

where: A is cross-sectional area, $i = \sqrt{I/A}$ is radius of inertia, and λ_i is slenderness ratio of the member.

Equation (3) is given by hyperbola function and it is valid until the critical stress is less than a proportionality limit, as it is shown in Figure 1. When the critical stress is exceeded, the member is buckling in the plastic range. Many scientists were dealing with this problem. Bauschinger first made an experimental study at the end of the nineteenth century. On the basis of this results and his own research, Tetmajer later suggested expression for the linear relation between stress and slenderness in the plastic domain:

$$\sigma_{cr} = \sigma_v \quad \text{for } 0 < \lambda < 60$$

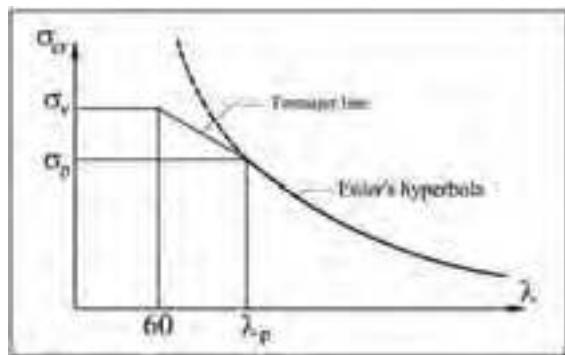
$$\sigma_{cr} = C_1 - C_2 \cdot \lambda \quad \text{for } 60 < \lambda < \lambda_p$$

where $\lambda_p = \pi \sqrt{E/\sigma_p}$ is the slenderness at the proportionality limit.

This linear function is also given in the Figure 1.

Engesser [9] also made the significant contribution by introducing the concept of the tangent modulus. Many other scientists also investigated these problems, as Karman and Shanley who modified Engessers curve. Some of the most significant buckling curves in the plastic domain are given in Figure 2.

Despite numerous experimental and theoretical studies, buckling problem, especially for the frames in the elastic-plastic domain, has not been completely solved. Fast development of computer technology has created the opportunity for a comprehensive solution of this problem. It should be mentioned that this problem recently was investigated by many authors whose results are presented, for example, in [24], [5] and [13].



Slika 1. Grafički prikaz Ojlerove hiperbole i Tetmajerove prave

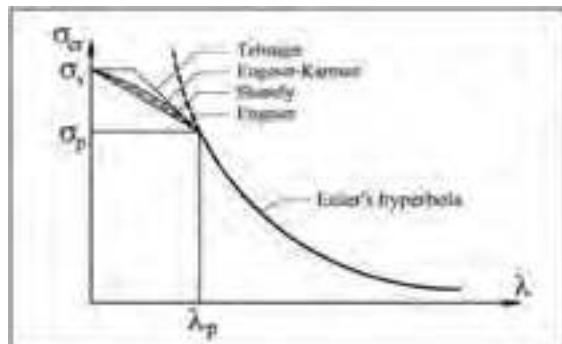
Figure 1. Graphical display of Euler's hyperbola and Tetmajer's line

U ovom radu je primenjena metoda konačnih elemenata kao najefikasnija numerička metoda za rešavanje stabilnosti okvirnih nosača. Kao što je dobro poznato, za određivanje kritičnog opterećenja primenom ove metode koristi se matrična jednačina u obliku:

$$\mathbf{K} \mathbf{q} = 0 \quad (4)$$

U okviru teorije elastične stabilnosti, matrična jednačina (4) rešava se inkrementalnim postupkom tako što se opterećenje povećava u zadatim inkrementima sve dok se ne dođe do kritične vrednosti iz uslova $\det \mathbf{K}=0$. Pri tome, u svakom štапу модул elastičnosti E има konstantну vrednost. Međutim, kod elasto-plastične analize postupak proračuna je komplikovaniji. Naime, pri svakom inkrementu opterećenja, u štapovima gde je pređena granica proporcionalnosti, mora da se promeni i krutost šтапа, odnosno koristi se novi tangentni modul E_t za taj štap. To znači i da su matrice krutosti koje se koriste u slučaju nelinearnog ponašanja materijala kompleksnije.

Da bi se sproveo proračun stabilnosti u neelastičnoj oblasti, potrebno je poznavati fizičko-mehaničke karakteristike materijala. Kao što je poznato, kada su u pitanju građevinski materijali poput čelika i betona, veza između napona i deformacije iznad granice proporcionalnosti postaje nelinearna. Na bazi eksperimentalnih istraživanja može se doći do ove zavisnosti, tj. dijagrama koji prikazuju vezu između napona i deformacija sve do nivoa naprezanja kada dolazi do iscrpljenja nosivosti materijala, odnosno loma nosećih elemenata konstrukcije. Tipičan dijagram, kada je u pitanju građevinski čelik, prikazan je na slici 3.



Slika 2. Krive izvijanja u plastičnoj oblasti

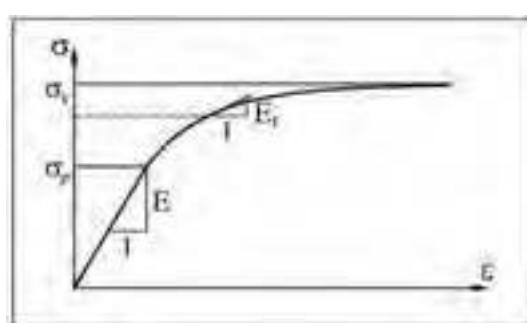
Figure 2. Buckling curves in the plastic domain

In this paper, the finite element method as the most efficient numerical method for solving such kind of problems is applied. As it is well known, using this method, the critical load can be obtained from the homogeneous matrix equation as the non-trivial solution:

$$\mathbf{K} \mathbf{q} = 0 \quad (4)$$

This problem can be solved by an incremental process, by increasing the load at the specified increments until the critical value is reached, i.e. until $\det \mathbf{K} = 0$. In the case of elastic stability problem, the modulus of elasticity E has a constant value. But, elastoplastic analysis is more complicated. For the structural member where the proportionality limit is exceeded, for each new load increment the member stiffness has to be changed and the corresponding tangent modulus E_t should be used for that member. It means that the stiffness matrices applied in the case of non-linear material behavior are more complex.

In order to implement the calculation of stability in inelastic range, it is necessary to know the physical and mechanical properties of materials. As it is well known, for the building materials (steel, concrete, ...) the relationship between stress and strain above the proportionality limit becomes nonlinear. On the basis of experimental results it is possible to obtain this diagram, which represents the relationship between stress and strain before the load bearing capacity is so reduced that the fracture of structure elements occurs. Typical stress-strain diagram of structural steel is given in Figure 3.



Slika 3. σ - ϵ dijagram čelika

Figure 3. Stress-strain diagram of structural steel

Ovaj dijagram pretstavlja vezu između napona σ i deformacije ϵ aksijalno pritisnutog štapa, gde je sa op obeležen napon na granici proporcionalnosti, a sa ov napon tečenja. Do granice proporcionalnosti modul elastičnosti E ima konstantnu vrednost i funkcija je samo vrste materijala. Sa daljim rastom opterećenja, ovaj modul postaje i funkcija nivoa naprezanja $E_t = f(\sigma)$ i naziva se tangentni modul [4]. Za razliku od E koji zavisi od karakteristika materijala, E_t je u funkciji i napona.

Na bazi eksperimentalnih istraživanja, jedna od najčešće korišćenih veza između ova dva modula, kada je u pitanju građevinski čelik, može se usvojiti u obliku [19], [8]:

This diagram shows the relationship between stress σ and strain ϵ of the axially loaded member, where op is the stress at the proportionality limit and ov is the yield stress. Below a proportionality limit, modulus of elasticity E is constant and it is function only of the material properties. Above this point, inelastic behavior occurs with a gradually decreasing resistance of material, measured by the tangent modulus E_t [4]. While E is only a function of the type of material, E_t is a stress dependent function.

On the basis of experimental research, one of the most used expressions that describe relationship between these two modules, for the structural steel, is given in the form [19], [8]:

$$E_t = 4E \cdot \left[\frac{\sigma}{\sigma_v} \left(1 - \frac{\sigma}{\sigma_v} \right) \right] \quad (5)$$

Ovo je empirijski izraz koji pokazuje ponašanje čeličnih stubova u neelastičnoj oblasti. Ova zavisnost je korišćena pri formiraju programu ALIN za nelinearnu elasto-plastičnu analizu okvirnih nosača.

Matrice krutosti kod nelinearnog ponašanja materijala imaju formalno isti oblik kao i pri linearном ponašanju materijala, ali se suštinski bitno razlikuju zato što se u svim članovima umesto ω javlja ω_t , a umesto konstantnog modula elastičnosti E javlja se tangentni modul E_t koji zavisi od nivoa naprezanja u elementu. Tako za slučaj štapa tipa „k”, koji je opterećen silom pritiska, imamo:

$$\mathbf{K} = \frac{EI}{l^3 \Delta_t} \begin{bmatrix} \omega_t^3 \sin \omega_t & \omega_t^2 l (1 - \cos \omega_t) \\ \omega_t l^2 (\sin \omega_t - \omega_t \cos \omega_t) & -\omega_t^2 l (1 - \cos \omega_t) \end{bmatrix} \text{ symm.}$$

gde je:

$$E_t = 4E \cdot \left[\frac{P_{cr,i}}{A \cdot \sigma_v} \cdot \left(1 - \frac{P_{cr,i}}{A \cdot \sigma_v} \right) \right] \quad (7)$$

$$\omega_t = \sqrt{\frac{P_{cr,i}}{E_t \cdot I}} \cdot l = \frac{1}{2} A \sigma_v l \cdot \sqrt{\frac{1}{EI(A\sigma_v - P_{cr,i})}} \quad (8)$$

$$\Delta_t = 2 \cdot (1 - \cos \omega_t) - \omega_t \cdot \sin \omega_t \quad (9)$$

Matrica krutosti za štap tipa „g”, kao i matrice krutosti za štapove koji su izloženi sili zatezanja prikazani su u [6].

3 PROGRAM ALIN ZA ANALIZU STABILNOSTI NOSAČA

Numerička analiza u ovom radu je obavljena primenom programa koji je napisan u C++ programskom

This is an empirical expression designed to represent the behavior of structural steel columns in the inelastic range. This expression was used in this analysis for developing program ALIN for the nonlinear elastic-plastic analysis of frame structures.

Stiffness matrix for nonlinear material behavior has the same form as for the linear behavior of the material, but they are essentially very different. Namely, the difference is primarily in the fact that constant modulus E is replaced by stress dependent tangent modulus E_t , and value ω is replaced by ω_t . So, for the member of the so-called type “k” (i.e. clamped at both ends), subjected to compressive force, stiffness matrix is:

$$\begin{bmatrix} -\omega_t^3 \sin \omega_t & \omega_t^2 l (1 - \cos \omega_t) \\ -\omega_t^2 l (1 - \cos \omega_t) & \omega_t l^2 (\omega_t - \sin \omega_t) \\ \omega_t^3 \sin \omega_t & -\omega_t^2 l (1 - \cos \omega_t) \\ \omega_t l^2 (\sin \omega_t - \omega_t \cos \omega_t) \end{bmatrix} \quad (6)$$

where:

Stiffness matrices of the member of the type “g” (i.e. hinged at one end and clamped at the other), and stiffness matrices for the members subjected to tension force are given in [6].

3 PROGRAM ALIN FOR STABILITY ANALYSIS OF FRAME STRUCTURES

The numerical analysis in this paper is performed using the code, developed in the C++ programming

jeziku. Program je nazvan ALIN i namenjen je kompleksnoj analizi linijskih nosača u ravni i prostoru. Osnovne mogućnosti ovog programa jesu analiza po teoriji prvog reda i linearizovanoj teoriji drugog reda (geometrijski nelinearna analiza), dinamička analiza, kao i analiza stabilnosti, odnosno proračun kritičnog opterećenja u elastičnoj i neelastičnoj oblasti. Detaljan opis i razvoj ovog programa pre svega je prikazan u [26], a takođe i u [6]. Treba istaći da se jedan od glavnih ciljeva ove analize sastoji u formiranju programa ALIN u delu koji omogućuje efikasno rešavanje problema stabilnosti u elasto-plastičnoj oblasti primenom „tačnih“ izraza za matrice krutosti koje su izvedene u prethodnom poglavljju.

4 ISTRAŽIVANJE STABILNOSTI OKVIRNIH NOSAČA PRIMENOM PROGRAMA ALIN

4.1 Ponašanje okvirnih konstrukcija u elasto-plastičnoj oblasti

Proračun stabilnosti okvirnih nosača u elasto-plastičnoj oblasti, zbog svoje složenosti, nije zastupljen u svakodnevnim inženjerskim proračunima. Zato se ni u postojećim standardima ne traži ovaj vid proračuna. Naime, određivanje kritične sile, odnosno kritičnog napona u plastičnoj oblasti za štapove okvirnog nosača obavlja se na bazi proračuna u elastičnoj oblasti i korišćenjem empirijskih izraza i krivih koje su dobijene na osnovu brojnih eksperimentalnih istraživanja ponašanja izolovanih štapova u plastičnoj oblasti. Ovde će se pokazati da se primenom programa ALIN može uspešno obaviti proračun stabilnosti okvirnih nosača i u elasto-plastičnoj oblasti. To s jedne strane omogućuje inženjerima da korišćenjem ovakvih kompjuterskih programa povećaju tačnost svojih proračuna i sagledaju stvarno ponašanje okvirnih nosača u elasto-plastičnoj oblasti. S druge strane, primena ovog načina proračuna treba da bude impuls za osavremenjivanje postojećih standarda u delu koji se odnosi na neelastično ponašanje ramovskih konstrukcija.

U nastavku je prikazan numerički primer šestospratnog okvira s tri polja (tj. četiri reda stubova), slika 4. Razmatran je slučaj uklještenog okvira s pomerljivim i nepomerljivim čvorovima. Analizirano je opterećenje koje deluje na svakom spratu, tj. slučaj kada se aksijalna sila u stubovima skokovito povećava gledajući odozgo ka dole.

Za okvirne nosače u ovoj analizi usvojeno je da su od čelika s karakteristikama: $E = 210,000,000 \text{ kN/m}^2$ i $\sigma_v = 240,000 \text{ kN/m}^2$. Iz izraza za tangentni modul (5) izračunava se vrednost napona na granici proporcionalnosti: $\sigma_p = 0.5 \cdot \sigma_v = 120,000 \text{ kN/m}^2$. Ovaj odnos napona u granici proporcionalnosti preporučuje se u [19], [8]. Za stubove i grede analiziranih okvira usvojeno je nekoliko različitih poprečnih preseka, pri čemu je uzeto da je $I_{\text{grede}} = 0.5 \cdot I_{\text{stuba}}$.

Na početku treba reći da su detaljni rezultati analize stabilnosti okvira s pomerljivim čvorovima (slika 4a) dati u [6]. U tabeli 1 prikazani su samo rezultati za kritično opterećenje za slučajeve različitih poprečnih preseka nosača na slici 4a i 4b.

language. The program is named ALIN and it enables the complex plane and space analysis of linear frames. The basic possibilities of this program are analysis according to the first and the second order theory, dynamic analysis and stability analysis, i.e. calculation of the critical load in the elastic and inelastic domains. The detailed description of this code can be found in [26], [6]. It should be emphasized that one of the main goals of this analysis was development of the part of program ALIN, which provides efficient solutions of stability problems in elastic-plastic domain. Those solutions are obtained using the "exact" expressions of the stiffness matrix which are presented in this paper.

4 INVESTIGATION OF THE STABILITY OF FRAME STRUCTURES USING THE PROGRAM ALIN

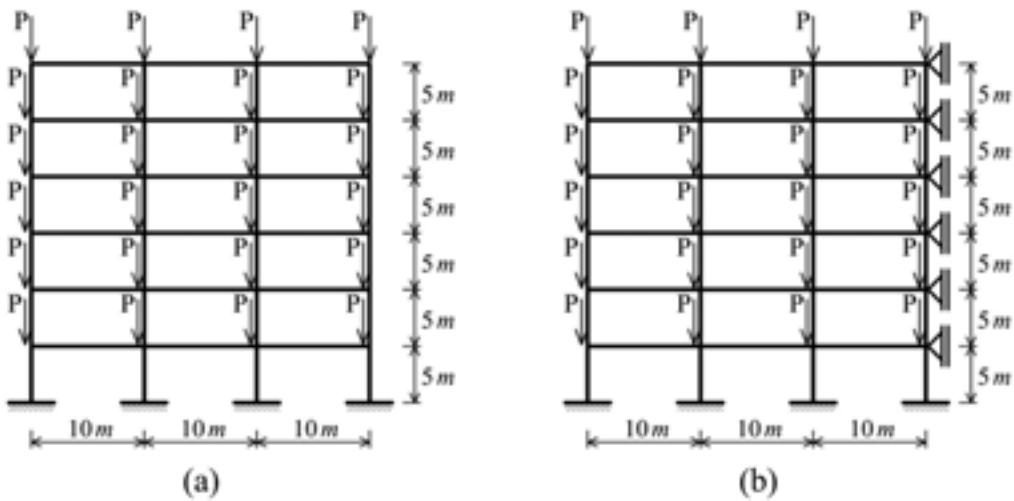
4.1 Behavior of the frame structures in elasto-plastic domain

Calculation of the stability of frames in the elasto-plastic domain, because of its complexity, is not used by engineers in the standard engineering stability analysis. It is the main reason why the current standards of frame stability analysis do not require this type of calculation. The determination of the critical load and the critical stress in the plastic field, for the analyzed members, usually is performed on the basis of the calculation in elastic domain and applying the empirical expressions and curves. That expressions and curves are obtained from a number of experimental studies related to the behavior of isolated members in the plastic field. In this paper it will be shown that calculation of the stability of frames in the elasto-plastic domain can be performed more successfully using the program ALIN. That allows engineers to use this computer program in order to increase the accuracy of their calculations and to consider the real behavior of the frame structures in elasto-plastic domain. On the other hand, the application of this numerical method may give the stimulus for the innovation of actual standards in the part related to the inelastic behavior of frame structures.

In the following, numerical example of six-story three-bay sway and non-sway frame is presented (Figure 4). The frame is clamped at the base, and concentrated load P is imposed on each column at each story. Since the axial force in columns is not constant, the elastic-plastic stability analysis can lead to the different behavior of the columns in the different floors.

In this analysis steel with characteristic: $E = 210,000,000 \text{ kN/m}^2$ and $\sigma_v = 240,000 \text{ kN/m}^2$ is used. From the tangent modulus expression (5), proportional limit is obtained as $\sigma_p = 0.5 \cdot \sigma_v = 120,000 \text{ kN/m}^2$ [19], [8]. Several different cross-sections are used for columns and girders of the analyzed frames, where it is assumed that $I_{\text{gird}} = 0.5 \cdot I_{\text{col}}$.

It should be noted that the detailed results of the stability analysis of the sway frame (Figure 4a) are given in [6]. Table 1 presents only the values of the critical load for all five analyzed cross-sections for both analyzed frames.



Slika 4. Numerički primer – šestospratni okvir s pomerljivim i nepomerljivim čvorovima
Figure 4. Numerical examples – six-story three-bay sway and non-sway frames

Tabela 1. Vrednosti kritične sile za okvire sa slike 4a i 4b - P_{cr} (kN)
Table 1. Values of critical load for the frames presented in Figure 4a and 4b - P_{cr} (kN)

	2[12]	2[16]	2[20]	2[26]	2[30]
Sl.4a Fig. 4a	$P_{cr,el}=26,01$	$P_{cr,el}=52,28$	$P_{cr,el}=96,41$	$P_{cr,inel}=209,39$	$P_{cr,inel}=307,58$
Sl.4b Fig. 4b	$P_{cr,inel}=113,67$	$P_{cr,inel}=176,35$	$P_{cr,inel}=243,19$	$P_{cr,inel}=372,07$	$P_{cr,inel}=457,51$

Iz navedenih rezultata može se zaključiti da se pomerljivi okviri većim delom izvijaju u elastičnoj oblasti. Međutim, za razliku od njih, sistemi s nepomerljivim čvorovima izvijaju se uglavnom u neelastičnoj oblasti. Tako se tokom proračuna u programu ALIN konstantna vrednost modula elastičnosti zamenjuje tangentnim modulom koji je u funkciji nivoa opterećenja u stubovima.

Vrednosti modula E i E_t u trenutku izvijanja prikazani su u tabeli 2. Jasno je da što su stubovi krući, oni primaju veće opterećenje, tako da su im vrednosti tangentnog modula manje. U tabeli 2 prikazani su rezultati za tri najopterećenije etaže zadatog okvirnog nosača.

Tabela 2. Vrednosti tangentnog modula za okvire sa slike 4b - E_t (kN/m²)
Table 2. Values of tangent modulus for the frame given in Figure 4b - E_t (kN/m²)

	3. sprat 3 rd floor	2. sprat 2 nd floor	1. sprat 1 st floor
2[8]	$E = 210,000,000$	$E_t = 208,233,739$	$E_t = 189,812,194$
2[12]	$E_t = 207,252,644$	$E_t = 177,569,958$	$E_t = 115,288,933$
2[16]	$E_t = 199,401,275$	$E_t = 150,828,451$	$E_t = 62,886,369$
2[20]	$E_t = 195,939,666$	$E_t = 140,944,179$	$E_t = 44,356,530$
2[26]	$E_t = 193,075,450$	$E_t = 133,169,928$	$E_t = 29,994,653$
2[30]	$E_t = 192,045,713$	$E_t = 130,444,186$	$E_t = 24,997,478$

This numerical example illustrates the difference in the stability analysis of braced and unbraced frames in the elastic-plastic field. In contrast to unbraced frames, braced frame structures buckle in inelastic domain. So, in the calculation using the code ALIN, constant modulus E is replaced by stress dependent tangent modulus E_t .

Values of the modulus E and E_t at the moment of buckling are given in Table 2. It is clear that as the columns are stiffer, they can be more loaded and their values of the tangent modulus are lower. Results for the three most loaded floors of the analyzed frames are given in Table 2.

4.2 Određivanje dužine izvijanja i nosivosti pritisnutih štapova okvirnih nosača

4.2.1 Određivanje dužine izvijanja pritisnutog štapa

Ispitivanje uzroka kolapsa pojedinih građevinskih konstrukcija, posebno kada su u pitanju čelične konstrukcije, pokazalo je da je često do toga dolazio usled loma pritisnutih elemenata konstrukcije. Ovi elementi doživeli su „prevremen“ lom tako da se i pre dostizanja dopuštenih napona iscrpela njihova nosivost. Može se generalno reći da kod najvećeg broja pritisnutih čeličnih štapova granična nosivost nije uslovljena kriterijumom nosivosti, već kriterijumom stabilnosti.

Proračun na bazi teorije elastične stabilnosti široko je primjenjen u inženjerskoj praksi, pošto se polazi od toga da se građevinske konstrukcije uglavnom ponašaju elastično kada su izložene svakodnevnim eksploracionim opterećenjima, pa i u slučaju kada opterećenje dostigne kritičnu vrednost. Naime, u želji da se projektuju vitke građevinske konstrukcije, često se dešava da se grade nosači velike visine i male krutosti, pa se izvijanje nosača dešava u elastičnoj oblasti. Zato je razumljivo da ovaj vid proračuna predstavlja osnovu standarda (propisa) za analizu stabilnosti okvirnih konstrukcija. On je definisan određivanjem tzv. efektivne dužine izvijanja pojedinih štapova okvirnih nosača.

Fizički gledano, dužina izvijanja je dužina zamenjujućeg obostrano zglobno oslonjenog štapa konstantnog preseka, opterećenog konstantnom normalnom silom. Kritična sila ima oblik:

i jednaka je kritičnoj sili posmatranog štapa proizvoljnih karakteristika. Matematički gledano, dužina izvijanja je rastojanje između susednih realnih ili fiktivnih prevojnih tačaka izvijenog štapa.

Dužina izvijanja prikazuje se pomoću proizvoda koeficijenta dužine izvijanja „ β “ i stvarne dužine štapa „ l “:

$$l_i = \beta \cdot l \quad (11)$$

Postupak određivanja efektivne dužine za štapove okvirnih konstrukcija prikazan je u JUS standardima [25], kao i evropskim normama EC3 za čelične konstrukcije [10], [11]. Na bazi tako određenih dužina izvijanja dalje se obavlja proračun aksijalno pritisnutih štapova korišćenjem tzv. krivih izvijanja. Na slici 5 prikazane su dužine izvijanja štapova s različitim uslovima oslanjanja. Na osnovu tako određene dužine izvijanja iz jednačine (10) nalazi se i veličina kritične sile.

4.2 Determination of the effective buckling length of the compressed members and determination of the load-bearing capacity of the compressed member

4.2.1 Determination of the effective buckling length of the compressed members

Investigation of the causes of the collapse of some building structures, especially made of steel, found out that collapse often occurred due to the failure of the compressed structural elements. It was found that failure of some elements happened in a way that before the allowable stress was reached, the member load-bearing capacity was exhausted. Generally it can be said that for the most of the compressed steel members, ultimate bearing capacity is determined by the stability criterion.

Calculation based on the theory of elastic stability is widely applied in engineering practice. It is because it can be assumed that building structures generally have elastic behavior when they are subjected to standard exploitation load, even in the case when the critical load is reached. Namely, in order to design structures with high slenderness, designers often make tall structures with low stiffness, so buckling occurs in the elastic domain. Therefore it is reasonable that this type of elastic calculation is the basis of the regulations for the stability analysis of frame structures. Such calculation procedure is defined through the determination of the effective buckling length of frame columns.

From the physical point of view, “effective buckling length” is a length of the equivalent member with constant cross-section that is pined at the both ends and is subjected to the compressive axial force. Critical force is defined by:

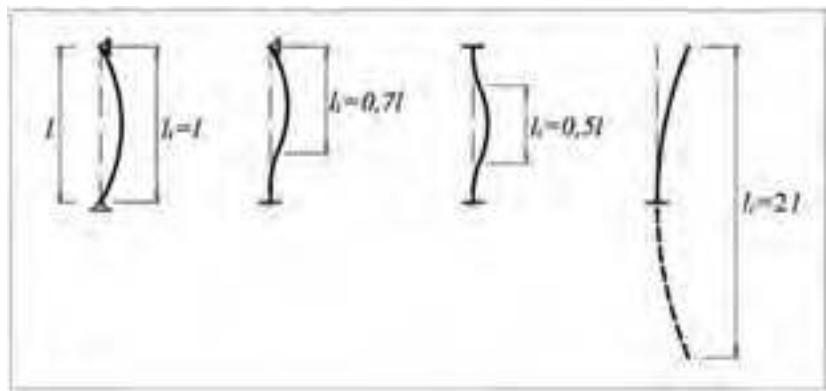
$$P_{cr} = \frac{\pi^2 EI}{(\beta \cdot l)^2} \quad (10)$$

and it is equal to the critical force of the analyzed member with arbitrary characteristics. From the mathematical (geometrical) point of view, “effective buckling length” is a distance between inflection points of the bended member.

The effective buckling length is given as the product of the column's effective length factor „ β “ and the geometric length of the column „ l “:

$$l_i = \beta \cdot l \quad (11)$$

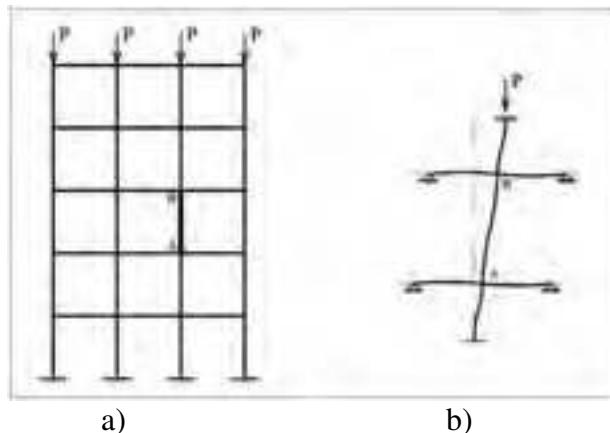
Procedure for calculation of the effective buckling length for the members in steel frame structures is given in national JUS standards [25] and European regulations EC3 [10], [11]. On the basis of such obtained buckling lengths, the calculation based on the buckling curves is applied for the axially compressed members. Effective buckling lengths for the members with different boundary conditions are shown in Figure 5. According to obtained buckling lengths, critical load is derived from the equation (10).



Slika 5. Dužina izvijanja štapova u funkciji uslova oslanjanja
Figure 5. Effective buckling lengths in function of the boundary conditions

Ovaj pristup proračuna kritične sile preko određivanja dužine izvijanja primjenjuje se i pri proračunu pritisnutih štapova okvirnih nosača [21]. Tako na primer, kada se vrši proračun stabilnosti okvirnog nosača sa slike 6a, koristi se izraz (10) i određuje dužina izvijanja, odnosno kritična sila svakog stuba ponaosob. Razlog za ovakav pristup leži u činjenici da se do nedavno smatralo da je proračun stabilnosti kompletne okvirne konstrukcije komplikovan za inženjersku praksu. Zato se u navedenim propisima za proračun stabilnosti okvirnih konstrukcija koriste uprošćene statičke šeme, kao na primer ona koja je prikazana na slici 6b. To praktično znači da se razmatra stub koji je „izdvojen“ iz okvira, a koji je elastično uklješten ili oslonjen samo na stubove i grede koji su u njegovoj neposrednoj okolini.

This approach to calculate the critical load, when the effective buckling length is obtained, is applied also to the columns of the frame structures [21]. For example, analysis of the whole frame structure (Figure 6a) is performed using the equation (10), and critical load and effective buckling lengths should be defined for each column individually. The reason for this is the fact that until recently it was considered that global stability analysis of frame structures is too complicated for engineering practice. That is the reason why the codes for the stability analysis of plane frame columns, use simplified static scheme, as presented in Figure 6b. Practically, this means that codes consider only columns which are isolated from the frame structure. These isolated columns are supported only by the adjacent columns and beams. Basically, presence of the other structural elements connected to the considered one is introduced by the corresponding boundary conditions.



Slika 6. Uprošćena statička šema za proračun stabilnosti prema propisima
Figure 6. Simplified static scheme for the stability analysis according to the codes

Ovakav približan, pojednostavljen proračun ima svojih prednosti jer se relativno lako dolazi do rezultata koji su prikazani u vidu odgovarajućih dijagrama i formula. Međutim, postavlja se pitanje da li su takva približna rešenja dovoljno tačna i da li se mogu primeniti na različite slučajevе koji se javljaju u inženjerskoj praksi. Postavlja se i pitanje da li ih i dalje treba koristiti, kada su usled burnog razvoja kompjuterske tehnike i

This approximate, simplified calculation has its advantages because results can be obtained very easily and they are shown by adequate diagrams and approximate formulas. However, very important question is how these approximate solutions are correct and how they can be used for the various examples in engineering practice. Also, there is important question should such solutions still be used, due to the fact that

programi za proračun stabilnosti okvirnih nosača postali opšte dostupni.

U ovom radu koristi se metodologija proračuna koja se zasniva na globalnoj analizi stabilnosti okvirne konstrukcije. To podrazumeva da se prvo odredi kritično optereće okvirnog nosača u celini $P_{cr,gl}$, a zatim se na osnovu njega određuju kritične sile za svaki pojedinačni stub P_{cr} . Na osnovu njih dobijaju se koeficijenti efektivne dužine izvijanja pojedinačnih stubova iz izraza:

$$\beta = \sqrt{\frac{\pi^2 EI}{P_{cr} \cdot l^2}} \quad (12)$$

U slučaju izvijanja u plastičnoj oblasti usvojiće se isti izraz za koeficijent β , s tim što se P_{cr} odnosi na kritičnu силу koja je sračunata na osnovu neelastičnog ponašanja okvira ($P_{cr,inel}$), a modul elastičnosti E više nije konstanta, već je funkcija nivoa naprezanja i zamjenjuje se tangentnim modulom E_t :

$$\beta = \sqrt{\frac{\pi^2 E_t I}{P_{cr,inel} \cdot l^2}} \quad (13)$$

4.2.2 Određivanje nosivosti pritisnutog štapa

Istraživanja prikazana u ovom radu zasnovana su na pretpostavkama o idealno pravom štalu od idealno homogenog materijala, i koji je idealno centrično opterećen. Jasno je da ove pretpostavke ne mogu biti u potpunosti ispunjene kada je u pitanju realan štap. Zato se od prvih eksperimenata (Bauschinger) pa sve do danas obavljaju intenzivna eksperimentalna istraživanja s ciljem da se što bolje sagleda stvarno ponašanje pritisnutih štapova.

Ova istraživanja ukazala su na brojne nesavršenosti realnih štapova koje izazivaju značajnu disperziju teorijskih i eksperimentalnih rezultata. Osnovni razlozi su mnogi faktori koji utiču na ponašanje i nosivost ovih štapova, kao što su: zaostali naponi, imperfekcije vezane za geometriju i oblik nosača, ekscentričnost pri opterećenju, struktura materijala, istorija opterećenja i drugo. Analizirajući pojedinačno ove faktore, može se konstatovati da, iako se polazi od pretpostavke da je materijal (čelik) idealno homogen, u praksi to nije slučaj. Ova nehomogenost materijala ima za posledicu različite module elastičnosti i granicu razvlačenja po poprečnom preseku nosača.

Jedan od vidova nesavršenosti (imperfekcija) materijala, osim njegove nehomogenosti, jeste i pojava sopstvenih (zaostalih) napona. Oni mogu biti izazvani termičkim uzrocima u toku proizvodnje kada se vrši valjanje ili hlađenje, kao i kasnije tokom procesa zavarivanja. Takođe, zaostali naponi mogu se javiti usled mehaničkih uzroka, kada se vrši ispravljanje limova i slično. Usled ovih zaostalih napona, tokom opterećenja (eksploatacije) dešava se kod centrično opterećenih štapova da se u pojedinim delovima poprečnog preseka formiraju plastične zone, iako središnji napon nije dostigao granicu razvlačenja. Zbog ove pojave smanjena je krutost štapa, pa samim tim i

fast development of computing possibilities is in the progress and the programs for stability analysis of frame structures become generally available.

The methodology of calculation that is used in this paper is related to the global stability analysis of frame structures. This means that critical load for the whole structure ($P_{cr,gl}$) should be calculated first. When this critical load is calculated, the critical load for each column (P_{cr}) can be obtained. Based on these results, effective length factor of individual columns is given by:

In the case of buckling in the plastic domain, the expression for the coefficient β has the same form as (12), but critical load (P_{cr}) is related to the critical force that is computed in the inelastic analysis of the frame ($P_{cr,inel}$). Also, modulus of elasticity E is no longer constant, but it is stress dependent and it is replaced by the tangent modulus E_t :

4.2.2 Determination of the load-bearing capacity of the compressed member

The investigations presented in this paper are based on the assumptions that the members are perfect, which means that they are perfectly straight and geometric imperfections are not taken into account. The load is also perfectly centric. It is clear that these assumptions can not be fully realized for the real members in engineering structures. Therefore, from the first experiments (Bauschinger) up to day, the intensive experimental research has been performed in order to predict the real behavior of compressed members.

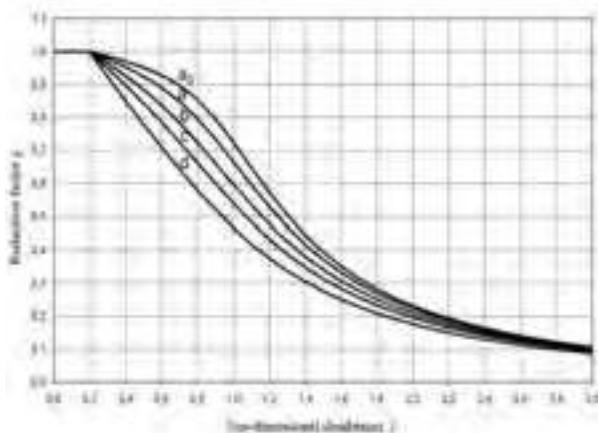
Such studies have indicated numerous imperfections of real members that lead to significant dispersion of theoretical and experimental results. These factors that have influence upon the behavior and load-bearing capacity of the observed members are: residual stresses, imperfections related to the geometry and shape of the girder, eccentric load, structure of the material, load histories, etc. Analyzing these factors separately, it can be concluded that, although it is assumed that the steel is ideal homogeneous material, it is not the case in practice. Such material inhomogeneity results in different moduli of elasticity and yield stress over the beam cross-section.

One of the types of material imperfections, besides its inhomogeneity is a phenomenon of residual stresses. They can be caused by thermal effects during production when mill rolling or cooling are performed, and later during the welding process. Also, residual stresses may occur due to mechanical causes, when straightening of steel sheet is performed. Due to these residual stresses, at some parts of the cross section of centrally loaded members, plastic zones might develop even though the normal stress has not reached the yield value. It reduces stiffness of the member, and therefore load bearing

njegova nosivost.

Realni nosači u građevinskim konstrukcijama izgrađeni su uvek s manjim ili većim geometrijskim nesavršenostima. Pri njihovoj montaži i formiranju građevinske konstrukcije dolazi do dodatnih imperfekcija ne samo u geometriji nosača već i u nanošenju opterećenja, odnosno pojavi neželjenih ekscentričnosti. Ove nesavršenosti predstavljaju početne deformacije pri izvijanju štapa, koje imaju za posledicu smanjenje granične nosivosti.

Kako bi se definisala stvarna nosivost centrično opterećenih štapova, prišlo se određivanju tzv. krivih izvijanja. Kao rezultat dugotrajnih eksperimentalnih i teorijskih istraživanja u razvijenim zemljama Evrope, došlo se do evropskih krivih izvijanja. Analitičkoj formulaciji ovih krivih najviše su doprineli Perry, Maquoi i Roundal [20], [18]. Ove krive našle su zatim primenu u standardima za proračun čeličnih konstrukcija gotovo svih evropskih zemalja. U evropskim normama one čine deo standarda dat u EC3 [11] i prikazane su na slici 7.



Slika 7. Krive izvijanja prema EC3 (preuzeto iz [11])
Figure 7. Buckling curves according to EC3 (taken from [11])

Naši propisi za proračun centrično opterećenih štapova od čelika takođe se zasnivaju na ovim istraživanjima i odgovaraju evropskim krivama iz standarda EC 3. Te krive su prikazane na slici 8.

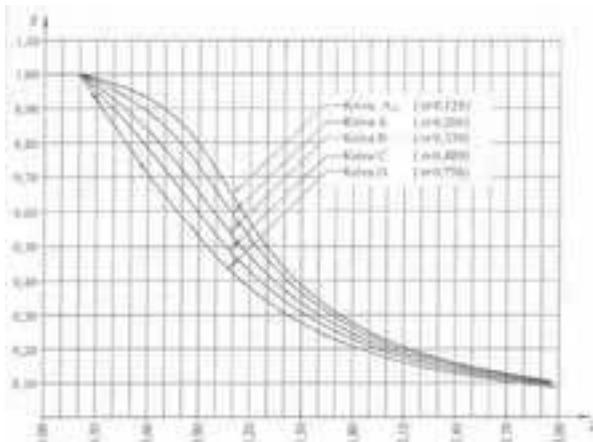
Ove krive izvijanja predstavljaju modifikaciju teorijskih krivih izvijanja, jer vode računa o svim imperfekcijama koje se javljaju kod realnih štapova. Pokazalo se da je nemoguće obaviti proveru nosivosti svih pritisnutih elemenata konstrukcije pomoću jedinstvene krive. Naime, mnogo je parametara koji utiču na izvijanja nosača, a to je pre svega početna deformacija štapa, oblik poprečnog preseka, nivo sopstvenih napona, način izrade i debljina delova poprečnog preseka. Da bi se obuhvatili svi ovi aspekti, neophodno je formirati seriju krivih izvijanja za svaki pojedinačni slučaj, kako je to definisano standardom.

Kako se sa slika 7 i 8 vidi, krive izvijanja definišu se preko bezdimenzionalnog koeficijenta otpornosti elementa na izvijanje χ , u funkciji relativne vitkosti $\bar{\lambda}$ i imperfekcija štapa. Naime, da bi se lakše vršio proračun, umesto uobičajene veze između kritičnog napona izvijanja σ_{cr} i vitkosti štapa λ uvodi se zavisnost u

capacity of the member is also reduced.

Real girders in civil engineering structures have been built with more or less geometrical imperfections. In assembly and construction of structures, additional imperfections may appear not only in geometry, but also when the load is applied and the unwanted eccentricity occurs. These imperfections represent the initial strain at buckling of the member and result in reduction of ultimate bearing capacity (critical load).

In order to define the real load bearing capacity of centrally loaded members, it was necessary to determine the so-called "buckling curves". As a result of long-term experimental and theoretical studies, European buckling curves are formulated. Analytical formulation of these curves was contributed most by Perry, Maquoi and Roundal [20], [18]. These curves then were applied in standards for design of steel structures of almost all European countries. In European regulations they are part of the standards given in EC3 [11] and are shown in Figure 7.



Slika 8. Krive izvijanja prema našim propisima (preuzeto iz [3])
Figure 8. Buckling curves according to JUS (taken from [3])

Our national standards for the calculation of centrally loaded steel members (JUS) are also based on these studies, and they are similar to European standards EC 3. These curves are given in Figure 8.

These curves represent a modification of the theoretical buckling curves because they take into account all imperfections that occur in real members. It has proved to be impossible to perform the calculation of load bearing capacity of all compressed structural elements using the single curve. Namely, a lot of parameters have influence on the buckling of girders, primarily: initial deformation of the member, the shape of the cross section, the level of residual stresses, method of fabrication of elements, the thickness of parts of the cross section and so on. In order to cover all these aspects it is necessary to define a series of buckling curves for each case, as it is defined in the standard.

As it can be seen from Figures 7 and 8, buckling curves are defined by reduction factor for relevant buckling mode χ that is the function of appropriate non-dimensional slenderness $\bar{\lambda}$ and imperfection factor of the member. Namely, in order to make calculations

bezdimenzionalnom obliku, preko koeficijenata χ i $\bar{\lambda}$. Koeficijent izvijanja χ dobija se kada se kritični napon izvijanja podeli s naponom na granici razvlačenja.

Da bi se očuvala zavisnost, relativna vitkost $\bar{\lambda}$ dobija se tako što se vitkost λ deli s vitkošću na granici razvlačenja:

$$\chi = \frac{\sigma_{cr}}{\sigma_y} \quad (14)$$

gde je:

$$\bar{\lambda} = \frac{\lambda}{\lambda_y} \quad (15)$$

Iz gornjeg izraza vidi se da vitkost na granici razvlačenja zavisi samo od vrste materijala. To znači da koeficijent χ za isti materijal direktno zavisi od vitkosti, odnosno dužine izvijanja štapa. Prema tome, da bi se ove krive izvijanja mogle ispravno koristiti, odnosno da bi davale tačne rezultate, neophodno je tačno odrediti vitkost, odnosno dužinu izvijanja štapa.

Treba napomenuti da su ove krive nastale na osnovu eksperimentalnih istraživanja ponašanja jednog izolovanog štapa s tačno definisanim graničnim uslovima, što je omogućilo i dobijanje tačne teorijske vrednosti kritične sile odnosno dužine izvijanja. Kako se ove krive koriste i za proračun stubova okvirnih nosača, potrebno je za svaki štap okvira tačno proračunati teorijsku vrednost kritične sile odnosno dužinu izvijanja.

Vec ranije je u ovom radu napomenuto da donedavno ovaj proračun nije bilo lako sprovesti, pa su u standardima data približna rešenja. U narednom delu rada dokumentovaće se da ta približna rešenja često prouzrokuju greške koje se ne mogu tolerisati. Kada se navedene krive izvijanja koriste za proračun štapova okvirnih nosača, posledica su greške u proračunu njihove nosivosti koje se takođe ne mogu tolerisati, jer se u proračun ulazi s netačnim rezultatima za dužinu izvijanja, odnosno vitkost štapa. Zato je u prethodnom delu rada data metodologija proračuna prema jednačini (12) koja definiše postupak za određivanja tačne vrednosti kritične sile, odnosno dužine izvijanja.

U nastavku će biti prikazani numerički primjeri koji ilustruju kakve greške nastaju pri proračunu štapova okvirnih nosača, kada se koriste približna rešenja iz postojećih standarda.

easier, instead of the usual relationship between the critical buckling stress σ_{cr} and member slenderness λ , the relationship in the non-dimensional form is introduced using the coefficients χ and $\bar{\lambda}$. Reduction factor χ is obtained when the critical buckling stress is divided by the stress at the yield point.

In order to use these curves, it is necessary that slenderness λ should be divided by the slenderness at the yield point:

$$\bar{\lambda} = \frac{\lambda}{\lambda_y} \quad (15)$$

where:

$$\lambda_y = \pi \sqrt{\frac{E}{\sigma_y}} \quad (16)$$

From the expression (16) it can be seen that slenderness at the yield point depends only on the material. It means that the coefficient χ for the same material is directly related to the slenderness, and respectively to the buckling length of the member. Therefore, in order that buckling curves could be used properly and that they would provide accurate results, it is necessary to accurately determine the slenderness and the buckling length of the member.

It should be mentioned that these curves were obtained on the basis of experimental studies of the behavior of an isolated member with clearly defined boundary conditions, so it is possible to calculate the exact theoretical value of the critical buckling load and buckling length. As these curves are used for calculation of columns of the frame structures, it is necessary to calculate accurately the theoretical value of the critical buckling load and buckling length for each member of the frame.

Previously, it was mentioned that until recently, this calculation was not easy to carry out, so the approximate solutions are given in the standards. The following section will present that these approximate solutions often lead to the errors that can not be tolerated. The consequence is that when these buckling curves are used for the calculation of members of frame structures, the substantial errors in the calculation of their load bearing capacity also occurs. The reason for this error is in the fact that calculation uses the incorrect results for the buckling length and slenderness of the member. Therefore, in the previous part of this paper the methodology of calculation according to the equation (12), which defines the procedure for determining the exact value of critical load and corresponding buckling length, is given.

The following section will present the numerical examples that illustrate what kinds of errors are arising in the calculation of members of frame structures, when the approximate solutions from the existing standards are used.

4.2.3 Numerički primer

U ovom delu analiziran je okvirni nosač sa slike 4b.

Kao što je ranije rečeno, u aktuelnim propisima prikazana su rešenja koja se dobijaju analizom elastične stabilnosti. Zato su ta rešenja i upoređena s rezultatima programa ALIN gde je dobijena „elastična” kritična sila, tj. pretpostavljeno je da je modul elastičnosti sve vreme konstantan. Međutim, kao što je već ranije pokazano, kod višespratnih okvirnih nosača s nepomerljivim čvorovima stubovi se često izvijaju u neelastičnom području. Zato je ovde sproveden i taj način proračuna.

Posmatran je primer gde su usvojene dimenzije elemenata kao i ranije: stubovi su visina $l=5m$, a grede dužina $2l=10m$. Karakteristike materijala su $E=210,000,000\text{ kN/m}^2$ i $\sigma_v=240,000 \text{ kN/m}^2$. Radi poređenja dobijenih rezultata, proračun je sproveden za svih šest do sada analiziranih poprečnih preseka. Vrednosti kritičnog opterećenja i odgovarajućeg tangentnog modula već su prikazane u tabelama 1 i 2. Prikaz rezultata koeficijenata efektivne dužine izvijanja za najopterećenije stubove (na prvoj etaži) dat je u tabeli 3.

*Tabela 3. Koeficijent β za stubove 1. sprata analiziranog okvira s nepomerljivim čvorovima
Table 3. Coefficient β for the columns on the first floor of the analyzed non-sway frame*

1.sprat 1 st floor	β (EC3)		β (jus)		β_{EL}	β_{INEL}
	unutr. inner	spolj. outer	unutr. inner	spolj. outer		
2L8	0.673	0.686	0.652	0.672	0.751	0.726
2L12						0.635
2L16						0.533
2L20						0.518
2L26						0.509
2L30						0.511

Prvo se može uočiti da kod stubova prvog najopterećenijeg sprata postoji solidno poklapanje rezultata dobijenih primenom propisa i primenom elastične analize, korišćenjem programa ALIN. Međutim, kao što je pokazano u [6], razlika u rezultatima se znatno uvećava kako se analiziraju stubovi na višim spratovima. Kao ilustracija toga, u tabeli 4 prikazane su vrednosti koeficijenta β za unutrašnje stubove analiziranog okvira, gde su rezultati primenom programa ALIN dati samo u slučaju analize u elastičnoj oblasti.

*Tabela 4. Koeficijent β za unutrašnje stubove okvira sa slike 4b
Table 4. Coefficient β for the inner columns of the frame given in Figure 4b*

metod proračuna calculat. method	ALIN (elast.)	EC3	differ. (%)	JUS	differ. (%)
sprat floor	1	0.751	0.673	-10.3	0.652
	2	0.823	0.906	10.0	0.857
	3	0.920	0.906	-1.5	0.857
	4	1.062	0.906	-14.7	0.857
	5	1.301	0.906	-30.3	0.857
	6	1.840	0.873	-52.5	0.816

4.2.3 Numerical example

The frame with characteristics given in Figure 4b is analyzed in this part.

As previously mentioned, the existing structural codes give solutions that are obtained using the elastic stability analysis. Therefore these solutions are compared with the results obtained using the program ALIN when the elastic critical force is calculated, i.e. when it is assumed that the modulus of elasticity E is constant. However, as already indicated before, non-sway multi-story frames often buckle in inelastic domain, so such kind of analysis also is performed herein.

Characteristics of the material and dimensions of the elements are taken the same as in the previous examples. Calculation is performed for all six considered cross sections. Results of the critical load and tangent modulus values are given in Tables 1 and 2. Table 3 represents results for the effective buckling length coefficient β for the most loaded columns (at the first floor).

First, it can be noticed that for the columns of the first floor there is a quite well coinciding of the results obtained using the code ALIN, as well as European and national codes for steel structures. However, as it is shown in [6], the differences in the results increase for the columns in the higher floors. As an illustration of this observation, Table 4 gives the values of the coefficient β for the inner columns of the analyzed frame. In this table, the results obtained by code ALIN are given only in case of elastic stability analysis.

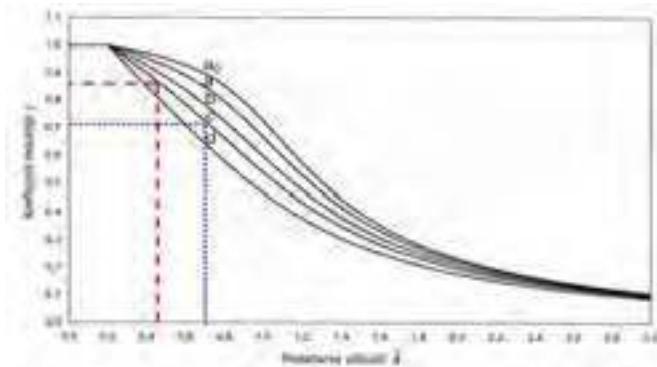
Razlog za ovakvu razliku u rezultatima je u tome što se proračun prema propisima bazira na analizi izolovanog elementa, tj. uzima se u obzir samo krutost stubova i greda koji su vezani u neposrednoj blizini krajnjih tačaka posmatranog elementa. Takođe, propisi ne uzimaju u obzir veličinu aksijalne sile u elementu.

Iz rezultata u tabeli 3, takođe se jasno vidi kolike se razlike dobijaju kada se proračun vrši u elastičnoj u odnosu na elasto-plastičnu oblast. Zato je u tabeli 5 data detaljnija analiza, odnosno prikazane su vrednosti koeficijenta β za stubove svih etaža ovog nepomerljivog šestospratnog okvirnog nosača.

*Tabela 5. Vrednosti koeficijenta β za stubove na svim spratovima analiziranog okvira
Table 5. Values of coefficient β for the columns in all stories of the analyzed frame*

	β_{el}	2[8]	2[12]	2[16]	2[20]	2[26]	2[30]
$\beta_{in,1}$	0.751	0.726	0.635	0.533	0.518	0.509	0.511
$\beta_{in,2}$	0.823	0.833	0.863	0.905	1.012	1.176	1.278
$\beta_{in,3}$	0.920	0.936	1.042	1.163	1.334	1.583	1.734
$\beta_{in,4}$	1.062	1.080	1.195	1.374	1.592	1.905	2.093
$\beta_{in,5}$	1.301	1.323	1.484	1.557	1.814	2.180	2.398
$\beta_{in,6}$	1.840	1.871	2.098	2.388	2.762	2.424	2.669

Na slici 9 prikazane su krive izvijanja za stubove prve etaže s poprečnim presekom 2[20 ($i=0.0589m$). Rešenja dobijena na osnovu elastične, odnosno neelastične analize prikazana su tačkastom, odnosno isprekidanom linijom.



Slika 9. Određivanje koeficijenta χ prema EC3 i JUS-u za 1.sprat analiziranog okvira s nepomerljivim čvorovima

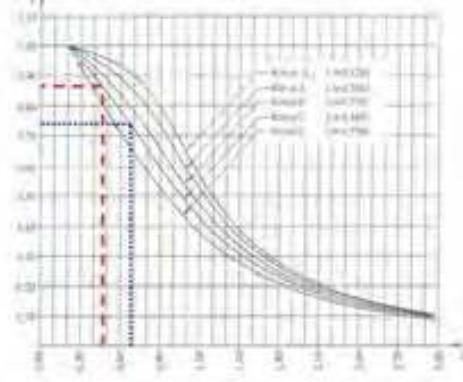
Figure 9. Determination of coefficient χ using the EC3 and JUS (first floor of the analyzed non-sway frame)

Očigledno je da se čini značajna greška u proračunu nosivosti stuba, ukoliko se umesto tačne vrednosti za β odnosno $\bar{\lambda}$ koje odgovaraju izvijanju u plastičnoj oblasti koriste vrednosti koje odgovaraju izvijanju u elastičnoj oblasti kako je to definisano postojećim standardima.

One of the main reasons for this difference in results is that the calculations according the codes are based on the isolated treatment of considered compressed elements. They only consider stiffness of the other structural elements connected to the considered one. Also, codes do not take into consideration the axial force value in the element.

Results from table 3 also shows what difference in results are obtained when the calculation is performed in elastic and elastic-plastic domain respectively. So, values of the coefficient β for the columns in all stories of the analyzed non-sway six story frame are given in Table 5.

Figure 9 represents the buckling curves for the columns of the first floor which have a cross-section 2[20 ($i=0.0589m$)). The solutions obtained on the basis of elastic and inelastic analysis are marked with dotted and dashed line, respectively.



It is obvious that significant error in the calculation of the load-bearing capacity of the compressed columns can be made. That happens when, instead of the exact values for β (and the corresponding $\bar{\lambda}$) that are obtained based upon the buckling in plastic domain, the results of elastic analysis are used, as it is defined in the existing structural codes.

5 ZAKLJUČAK

U ovom radu predložen je proračun koji se zasniva na analizi globalne stabilnosti okvirne konstrukcije i – u prvom koraku – određivanju kritične sile za konstrukciju u celini. Zatim se u nastavku određuju kritične sile, odnosno dužina izvijanja pojedinih pritisnutih štapova i to na bazi odnosa između ukupne kritične sile u konstrukciji i normalne sile u pojedinim štapovima.

Predloženi postupak nije nepoznat, ali način kako je on ovde formulisan i sproveden nije do sada primenjen ni u jednom od komercijalnih programa koji se bave stabilnošću okvirnih nosača. Tako se za interpolacione funkcije usvajaju rešenja koja se dobijaju iz diferencijalne jednačine savijanja štapa po teoriji drugog reda. U ovom slučaju je dovoljno da se usvoji samo po jedan konačni element duž svakog štapa u okvirnoj konstrukciji, čime se drastično smanjuje broj nepoznatih, odnosno jednačina u metodi konačnih elemenata. Upravo je i to glavna prednost ovog postupka proračuna i formiranog programa ALIN u odnosu na uobičajeni postupak koji se zasniva na primeni geometrijske matrice krutosti.

U radu je ispitivana tačnost rešenja koja su data u našim JUS i EC3 standardima, a odnose se na dužine izvijanja pritisnutih štapova okvirnih nosača. Primenom predložene metode pokazano je da se u pojedinim slučajevima čine greške ako se koriste postojeći izrazi i dijagrami dati u našim standardima. To znači da je potrebno prići inovaciji ovih standarda na način kako je to, na primer, ovde predloženo, kako bi se inženjerima u praksi omogućio tačniji način proračuna. U tom smislu već su učinjeni koraci za izmenu evropskih standarda (EC3), posebno u delu koji se odnosi na složene deformabilne konstrukcije gde se zahteva proračun po teoriji drugog reda, ali bez navođenja detalja tog proračuna.

Predloženi postupak proračuna u ovom radu korišćen je i za elasto-plastičnu analizu kada se u proračun osim geometrijske uvodi i materijalna nelinearnost. Izvedene su matrice krutosti korišćenjem tangentnog modula elastičnosti koji prati promenu krutosti štapa u neelastičnoj oblasti. Treba istaći da se u napred navedenim standardima kritična sila u plastičnoj oblasti određuje samo približnim proračunom. On se sastoji u tome da se prvo odredi kritična sila kao da se štap izvija u elastičnoj oblasti, a zatim se korišćenjem krivih izvijanja koje su u standardima definisane preko približnih (empirijskih) izraza, određuje kritična sila u plastičnoj oblasti.

Doprinos ove analize sastoji se u tome što se na osnovu formiranog algoritma proračuna koji je implementiran u programu ALIN, može analizirati ponašanje okvirne konstrukcije, ne samo u elastičnoj već i plastičnoj oblasti. Time je omogućeno praćenje fenomena gubitka stabilnosti okvirne konstrukcije i u plastičnoj oblasti i direktno određivanje kritične sile pri njenom kolapsu.

5 CONCLUSIONS

In this paper the global stability analysis of whole frame structure is suggested, and in first step, determination of the critical load for the structure as a whole. Then, the critical load and effective buckling length of each member can be found.

The proposed method is not unknown, but the way how it is formulated and implemented here has not been applied in any of the commercial programs that deal with the stability of frames. The calculation where interpolation functions are derived from the exact solution of the differential equation of bending according to the second order theory is proposed. The advantage of such an approach, which is applied in the program ALIN, is in the fact that only one finite element is needed for each beam or column, so the total number of finite elements is significantly less than in the usual approach based on the geometric stiffness matrix.

The accuracy of the solutions given in JUS and EC3 standards, related to the effective buckling length determination was investigated. Applying the proposed method it was shown that some errors are made when the approximate solutions from the codes are used. That means that there is a need for the innovation of these standards in the part where the effective length of frame columns is considered. It should be emphasized that some steps have already been taken in EC3 standards, in the part related to the complex deformable structures which require the calculation according to second order theory, however, no more details are given in EC3.

The proposed calculation method has been used also for the elasto-plastic analysis when the geometrically non-linear process is followed by development of the material nonlinearity as well. Stiffness matrices are derived using the tangent modulus which is stress dependent and that follows changes of the member stiffness in the inelastic field. These matrices have been implemented in the computer code ALIN. It should be mentioned that in the analyzed standards, a critical force in the plastic domain is determined by the approximate calculation. First, the critical load is calculated when the buckling occurs in elastic domain. Then, using the buckling curves that are defined through the approximate (empirical) expressions, the critical load in plastic domain is determined.

The contribution of this analysis is based upon the calculation algorithm implemented in code ALIN. This algorithm introduces more accurate calculation of buckling not only in elastic, but also in the plastic domain. It allows monitoring of the phenomena of stability loss of the frame structure in the plastic domain and direct determination of the critical force at the moment of buckling.

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REZIME

NELINEARNA ANALIZA STABILNOSTI OKVIRNIH NOSAČA

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U ovom radu istraživan je fenomen gubitka stabilnosti okvirnih nosača u elasto-plastičnoj oblasti. Numerička analiza je sprovedena primenom metode konačnih elemenata. Matrice krutosti su izvedene korišćenjem trigonometrijskih interpolacionih funkcija koje se odnose na tačno rešenje diferencijalne jednačine savijanja štapa prema teoriji drugog reda. U slučaju kada se izvijanje konstrukcije dešava u plastičnoj oblasti, konstantan modul elastičnosti E u matrici krutosti zamenjen je tangentnim modulom E_t koji prati promenu krutosti štapa u neelastičnoj oblasti i funkcija je nivoa opterećenja u štapu. Za potrebe ove analize formiran je deo računarskog programa ALIN koji može da se koristi za elastičnu i elasto-plastičnu analizu stabilnosti okvirnih konstrukcija. Program je napisan u C++ programskom jeziku. Primenom ovog programa omogućeno je i određivanje kritičnog opterećenja okvirnih nosača u elastičnoj i neelastičnoj oblasti. U ovom istraživanju formiran je i algoritam za proračun dužina izvijanja pritisnutih štapova stubova okvirnih nosača, a koji se bazira na proračunu globalne analize stabilnosti okvirne konstrukcije. Rezultati dobijeni primenom ovog algoritma upoređeni su s rešenjima koja se dobijaju korišćenjem evropskih EC3 i domaćih JUS standarda za okvirne čelične konstrukcije, a koja su približnog karaktera. Na osnovu postupka koji je dat u ovom radu moguće je praćenje fenomena gubitka stabilnosti okvirnog nosača u plastičnoj oblasti i direktno određivanje njegove kritične sile u toj oblasti.

Ključne reči: stabilnost konstrukcija, okvirni nosači, nelinearna analiza, metod konačnih elemenata, tangentni modul

SUMMARY

NONLINEAR STABILITY ANALYSIS OF THE FRAME STRUCTURES

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In this paper the phenomenon of instability of frames in elasto-plastic domain was investigated. Numerical analysis was performed by the finite element method. Stiffness matrices were derived using the trigonometric shape functions related to exact solution of the differential equation of bending according to the second order theory. When the buckling of structure occurs in plastic domain, it is necessary to replace the constant modulus of elasticity E with the tangent modulus E_t . Tangent modulus is stress dependent function and takes into account the changes of the member stiffness in the inelastic range. For the purposes of numerical investigation in this analysis, part of the computer program ALIN was created in a way that this program now can be used for elastic and elasto-plastic stability analysis of frame structures. This program is developed in the C++ programming language. Using this program, it is possible to calculate the critical load of frames in the elastic and inelastic domain. In this analysis, the algorithm for the calculation of buckling lengths of compressed columns of the frames was also established. The algorithm is based on the calculation of the global stability analysis of frame structures. Results obtained using this algorithm were compared with the approximate solutions from the European (EC3) and national (JUS) standards for the steel structures. By the given procedure in this paper it is possible to follow the behavior of the plane frames in plastic domain and to calculate the real critical load in that domain.

Key words: stability of structures, frame structures, nonlinear analysis, finite element method, tangent modulus