

Strategic Design of Wall Envelopes for the Enhancement of Building Thermal Performance at Reduced Air-Conditioning Costs

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Abstract

A strategy is proposed for the design of wall envelopes to improve unsteady thermal performance parameters in non-air-conditioned buildings and to reduce energy costs in air-conditioned buildings. The thermophysical properties of building materials (e.g., burnt bricks, mud bricks, laterite stone, cinder concrete, and expanded polystyrene) were measured experimentally using a thermal analyzer. The total of 28 combinations of composite walls were designed with expanded polystyrene as an insulation material based on seven criteria and were subjected to 8 different external surface heat transfer coefficient, which were tested for unsteady thermal performance parameters and air-conditioning cost-saving potential. In this paper, unsteady thermal transmittance obtained from admittance method has been employed to compute cost saving potential of air-conditioning for the various wall envelopes. The use of C-H₅ design at a 2 m/s wind speed increases the decrement lag of burnt brick, mud brick, laterite stone, and cinder concrete composite wall envelopes by 48.1%, 49.0%, 59.5%, and 47.0%, respectively, relative to the common wall design (C-H₁) in non-air-conditioned buildings. The laterite with a C-H₅ design offers the highest annual energy cost savings (1.71 \$/m² at 2 m/s), highest life cycle cost savings (18.32 \$/m² at 2m/s), and the lowest payback period (4.03 yrs at 2 m/s) as compared to the other

29 building materials in air-conditioned buildings. The overall results of this study are expected to open
30 new paths to simple design considerations for energy-efficient buildings.

31

32 **Keywords:** Energy-efficient wall design; Attenuation factor; Decrement lag; Annual energy; Life cycle

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35 **1. Introduction**

36 Building materials serve as thermal mass in passive building designs. Heat transfer takes place at the
37 air and building wall interface by both radiation and convection. The convective heat transfer at the air
38 and building wall interface depends on convective heat transfer coefficients. These coefficients are
39 adversely influenced by the atmospheric wind velocity; hence, the stability of the thermal characteristics
40 of building walls is affected by changes in the atmospheric wind velocity (Davies, 2004).

41 The amplitude of the sinusoidal heatwaves decreases as they penetrate through a building envelope.
42 This reduction is due to the thermal mass of the building material used for the envelope. The shrinkage
43 of the heatwave from the outside to the inside of the building envelope is known as the attenuation
44 factor. The time taken for the heat sine wave to penetrate through the building envelope is known as
45 decrement delay (Duffin, 1984). Building enclosure materials should have low attenuation factors and
46 high phase shift values to maintain comfortable indoor conditions (ASHRAE, 2001). Consequently,
47 much effort has been made to evaluate the influence of building enclosure thickness and insulation
48 location (e.g., within the enclosure) on variables such as phase lag or thermal delay and attenuation
49 factor using the Crank–Nicolson (C-N) procedure (Antonopoulos and Koronaki, 2000; Lee Ok et al.,
50 2014). The admittance method has been used to find the surface factor of building enclosures (Evola and
51 Marletta, 2013), and can be used to assess the thermal characteristics of roofs and wall envelopes (Hall

52 M R, 2010; Najim, 2014; Najim and Fadhil, 2015; Shaik and Talanki, 2016). The effect of the
53 thermophysical properties of wall enclosures on the decrement delay and attenuation factor has also
54 been explored (Asan, H., 1998; Kontoleon et al., 2013; Ulgen, 2002). Atmospheric moisture and
55 temperature can also affect the attenuation factor and decrement delay of laterite building wall
56 enclosures (Shaik and Talanki Puttaranga Setty, 2016). A study of south-facing walls in the
57 Mediterranean region indicated that a 0.2 increase in solar absorptivity increases the decrement delay;
58 however, further increases in solar absorptivity decreased decrement delay (Kontoleon and
59 Eumorfopoulou, 2008). The implicit finite difference procedure was adopted to compute attenuation
60 factors and attenuation delays in Turkey (Ozel and Pihtili, 2007). The admittance method was
61 customized to incorporate moisture-dependent unsteady parameters (Hall and Allinson, 2008). The
62 attenuation factor and decrement delay of AAC concrete were studied (Ng et al., 2011), showing an
63 increase in the thermal diffusivity of the wall enclosure materials increased the attenuation factor and
64 decreased the decrement delay. Decrement delay and attenuation factor values have been computed by
65 explicit control volume discretization (Mavromatidis et al., 2012) and the finite difference method. It
66 has been suggested that decrement delay values could be affected by the azimuth angle of the wall
67 enclosure (Ruivo et al., 2013). Moreover, a coconut fiber-filled, Ferro-cement prototype house was also
68 found to offer higher resistance to heat flow at peak times compared to concrete slab roofing in Mexico
69 (Alavez-Ramirez et al., 2014). Thermal performance studies of ventilated roofs showed that the best
70 performance could be obtained by placing insulation close to the cold surface and below the air space
71 layer (Gagliano et al., 2012). Various building envelope configurations were investigated for thermal
72 performance using finite difference and admittance methods (Balaji et al., 2019). Expanded polystyrene
73 was reported to be the best insulating material with a maximum life cycle cost savings and minimum
74 payback period (Yu et al., 2009). A thermoeconomic method of optimizing the insulation thickness gave

75 better results compared to the entransy-based environmental method (Gülten, 2020). The thickness of
76 phase change materials and the insulation layer were simulated and optimized with simulation tools and
77 generic algorithms (Baniassadi et al., 2016). It was concluded that the insulation layer is more effective
78 than the phase change materials in moderate climates. The mathematical model for evaluating the
79 economic and life cycle cost savings of insulation in the non-homogenous building walls was developed
80 and studied (Huang et al., 2020). The Lagrangian optimization method was used to optimize the
81 insulation thickness with the maximum life cycle cost savings and minimum payback period (Feng et
82 al., 2018). The exergo-economic method was carried out to optimize the insulation thickness in a
83 method where combustion parameters and insulation cost are considered (Arslan et al., 2010).

84 Currently, there is little information on the design of wall envelopes to reduce heat gain by
85 convection and various wall design energies and cost assessments – especially considering low-cost
86 building options in emerging economies. In this respect, we investigated the optimum design conditions
87 for building wall envelopes to facilitate comfortable indoor conditions in hot climatic environments
88 based on the thermal performance parameters of the walls. The thermal parameters considered in this
89 study included: admittance, unsteady transmittance, attenuation factor, and decrement delay. The
90 energy-economic parameters studied include annual energy cost savings, life cycle cost savings, and
91 payback periods. The building envelope is a heat flow control element for the building. The appropriate
92 design of the envelope reduces the building cooling loads. Heat transfer from the outer to the inner layer
93 of the envelope depends mainly on the outside surface heat transfer coefficients. Therefore, in this
94 paper, we proposed optimum envelope designs for reduced cooling loads under various external heat
95 transfer coefficients. The reliability of the results has been validated against the Chartered Institution of
96 Building Services Engineers guide (CIBSE, 2006).

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2. Materials and methodology

2.1 Materials

105 This study considered the most widely used building materials in India (i.e., burnt brick, mud brick,
106 laterite stone, and cinder concrete). Burnt bricks are composed of clay (20-30% by mass), silt (20-35%
107 by mass), and sand (35-50% by mass) (IS: 2117, 2002). The total content of clay, silt, and water should
108 not be less than 50%. The molded bricks are fired in a kiln at temperatures ranging from 900-1000°C to
109 obtain burnt bricks. The compressive strength of burnt bricks should not be less than 3.5 N/mm² as per
110 Indian standards. Mud bricks or adobe are made of clay (15% by mass), silt (10-30% by mass), and sand
111 (50-70% by mass) mixed with straw, and they are air-dried before use. The compressive strength of mud
112 bricks should not be less than 2 N/mm² for building applications as per Indian standard IS: 1725-1982
113 (IS: 1725, 1982). Laterite stone is abundantly available on the south-west coast of India. It is a readily
114 available, eco-friendly, economical, and ferruginous building material. The compressive strength of
115 laterite stone should not be less than 3.5 N/mm² for building enclosures (IS: 3620, 1979). Cinder
116 concrete contains cement, sand, and gravel in a 1:2:3 ratio. Cinder concrete replaces 20% of the Portland
117 cement with fly ash (IS: 6042, 1969). Its compressive strength should not be less than 2 N/mm².
118 Expanded polystyrene was used as an insulation material between the wall layers. Cement plaster
119 (cement to sand mortar ratio of 1:6) was applied at either side of the wall envelope for all material types.

2.2 Experimental Methodology

121 Fig. 1 presents images of the building and insulating materials of the wall enclosures considered. In
122 this paper, these materials were used as building enclosure materials for analyzing thermal performance.

123 Thermal properties such as specific heat capacity and thermal conductivity were measured
124 experimentally using the KD2 Pro thermal properties analyzer (hot wire probe method) (ASTM:D5334-
125 14, 2016). The above equipment can measure thermal conductivity in a range from 0.02 to 2.00 W/mK
126 and volumetric specific heat from 0.5 to 4.0 MJ/m³K with 10% accuracy. The probe consists of a dual
127 needle (30 mm length, 1.3 mm diameter, and 6 mm spacing). The transient heat source has supplied
128 from an electrical input. One needle acted as a heating source, and the other acted as a monitoring
129 source. The thermophysical properties were obtained from the temperature-time relationship. Two holes
130 were drilled in building materials with the corresponding size so that the probe needle fit tightly into the
131 holes. During the measurement, thermal grease was applied to the dual needle to avoid contact
132 resistance error. The density of the brick was obtained by the ratio between its average mass and volume
133 (IS:2185, 2005). The uncertainty of thermal conductivity, density, and specific heat were calculated for
134 each of all tested building materials (Holman, 2012). The measured thermophysical properties of
135 building materials with uncertainty are presented in Table 1.

136
137 **Fig. 1.** Building and insulation materials of wall enclosures.

138
139 **Table 1** Thermophysical properties of building wall enclosure materials.

140 141 **2.3 Thermal analysis:**

142 Seven different wall envelope configurations with four brick materials with either side cement
143 plastered (burnt brick, mud brick, laterite stone, and cinder concrete) and one insulation material
144 (expanded polystyrene) were considered for the investigation of dynamic thermal characteristics.
145 Thermal performance was analyzed under various atmospheric wind velocities. The configurations

146 investigated include: (1) no insulation (C-H₁), (2) with an insulation layer at the external side (C-H₂), (3)
147 with an insulation layer at the center (C-H₃), (4) with an insulation layer on the internal side (C-H₄), (5)
148 with one insulation layer on the external side and another at the internal side (C-H₅), (6) with one
149 insulation layer at the external side and another at the center (C-H₆), and (7) with one insulation layer at
150 the center and another at the internal side of the envelope (C-H₇). Fig. 2 shows the configurations of the
151 composite wall enclosures and the expanded polystyrene insulation considered.

152

153 **Fig. 2.** Configuration of composite wall enclosures and expanded polystyrene insulation considered in
154 this study.

155

156 Information related to the dynamic characteristics of the wall envelopes (as many as 28
157 combinations of composite walls) was computed and investigated to identify the best wall envelope
158 configuration for reduced cooling loads under various atmospheric wind velocities. The thermal
159 admittance method was employed to compute the dynamic thermal characteristics of the wall envelopes.
160 The admittance procedure uses matrices to simplify the temperature and energy cycles for a composite
161 building fabric enclosure (wall or roof) that is subjected to sinusoidal temperature variations at the sun-
162 air node of the enclosure. The building walls do not generate any internal heat, and their thermal
163 properties are the same in all three directions. The governing diffusion equation for the three dimensions
164 of the wall for temperature $T(x,y,z,t)$ without internal heat generation and the same thermal properties in
165 three dimensions can be written as Eq. (1).

$$\frac{\partial T}{\partial t} = \frac{k}{\rho C p} \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right) \quad (1)$$

166

167 The building wall has three dimensions (i.e., length, breadth (thickness), and height). Since the
168 temperature differences along the length (y) and height of the wall (z) are small, the diffusion equation is
169 reduced to one dimension (i.e., through the thickness (x) of the wall). The homogeneous and composite
170 walls are exposed to periodic cyclic variations in temperature with heat flow through the wall enclosure
171 in one direction through the thickness of the wall (i.e., horizontally).

172 The diffusion equation for heat transfer through the thickness of the wall is presented in Eq. (2) (Davies,
173 2004).

$$\frac{\partial^2 T(x, t)}{\partial X^2} = \frac{\rho C p}{k} \frac{\partial T(x, t)}{\partial t} \quad (2)$$

174

175 The periodic solution of the diffusion equation is presented in Eqs. (3) to (7).

176

$$T(x, t) = A \exp(x/\xi) \exp(t/\zeta) \quad (3)$$

177

178 Eq. (3) should satisfy the Fourier equation, which is possible only if $\zeta^2 = \alpha \xi$.

179 Here,

$$\alpha = \frac{k}{\rho C p} \quad (3a)$$

180

$$\zeta = \frac{P}{j2\pi} \quad (3b)$$

181

$$\xi = \sqrt{\frac{\alpha P}{j2\pi}} \quad (3c)$$

182

$$j = \sqrt{-1} \quad (3d)$$

183
 184 The periodic solution for a wall enclosure of finite thickness exposed to sinusoidal excitation with a
 185 given period can be obtained by imposing boundary conditions. The equation can be obtained as
 186 follows,

$$\frac{x}{\xi} = \frac{x}{\pm(\alpha P / j 2\pi)^{1/2}} = \pm(1+j) \left(\frac{\pi \rho c_p x^2}{kP} \right)^{1/2} = \pm(1+j)\gamma_1 x \quad (4)$$

187 where

$$\gamma_1 = \sqrt{\pi \rho c_p / kP} \quad (4a)$$

188
 189 The temperature can be represented as follows:

$$T(x, t) = [Q \sinh(\gamma_1 x + j\gamma_1 x) + R \cosh(\gamma_1 x + j\gamma_1 x)] \exp(j2\pi t / P) \quad (5)$$

190

$$T_{ext} = T_{int} \cosh(u + ju) + q_{int} (\sinh(u + ju)) / a \quad (5a)$$

191

$$q_{ext} = T_{int} (\sinh(u + ju)) \times a + q_{int} \cosh(u + ju) \quad (5b)$$

192

193 The above equations can be rearranged, as shown in Eq. (6).

$$\begin{bmatrix} T_{ext} \\ q_{ext} \end{bmatrix} = \begin{bmatrix} \cosh(u + ju) & (\sinh(u + ju)) / a \\ (\sinh(u + ju)) \times a & \cosh(u + ju) \end{bmatrix} \begin{bmatrix} T_{int} \\ q_{int} \end{bmatrix} \quad (6)$$

194 T_{ext} and q_{ext} are sinusoidally varying temperature and heat flux of complex values, respectively, at the
 195 external side of the wall enclosure. The values q_{int} and T_{int} are at the internal side of the wall enclosure.

196 Here,

$$u = \sqrt{\frac{\pi \rho c_p X^2}{kP}} \quad (6a)$$

197

$$a = \sqrt{\frac{j2\pi k \rho c_p}{P}} \quad (6b)$$

198

199 A homogeneous wall enclosure transmission matrix can be written as Eq. (7),

$$\begin{bmatrix} A_1 + jA_2 & (A_3 + jA_4)/a \\ (-A_4 + jA_3).a & A_1 + jA_2 \end{bmatrix} \quad (7)$$

200 where

$$A_1 = \cosh(u) \cos(u) \quad (7a)$$

201

$$A_2 = \sinh(u) \sin(u) \quad (7b)$$

202

$$A_3 = [\cosh(u) \sin(u) + \sinh(u) \cos(u)]/\sqrt{2} \quad (7c)$$

203

$$A_4 = [\cosh(u) \sin(u) - \sinh(u) \cos(u)]/\sqrt{2} \quad (7d)$$

204

205 The convective heat transfer coefficient at the external surface as per CIBSE standards.

$$h_{\text{sext}} = 5.8 + 4.1C_s \quad (\text{For all wind speeds}) \quad (7e)$$

206

$$h_{\text{sext}} = 16.7 C_s^{0.5} \quad (\text{For wind speeds up to 3.5 m/s}) \quad (7f)$$

207

$$h_{sint} = 2.5 \text{ W/m}^2 \text{ K (For internal surface resistance)} \quad (7g)$$

208

209 The homogeneous wall enclosure transmission matrix can be written as Eq. (8)[39],

$$\begin{bmatrix} A_1 + jA_2 & (A_3 + jA_4)/a \\ (-A_4 + jA_3).a & A_1 + jA_2 \end{bmatrix} \quad (8)$$

210 where,

$$211 \quad A_1 = \cosh(u) \cos(u),$$

$$212 \quad A_2 = \sinh(u) \sin(u),$$

$$213 \quad A_3 = [\cosh(u) \sin(u) + \sinh(u) \cos(u)]/\sqrt{2},$$

$$214 \quad A_4 = [\cosh(u) \sin(u) - \sinh(u) \cos(u)]/\sqrt{2}.$$

215 The matrices for internal (R_{sint}) and external (R_{sext}) surface film resistances for the wall enclosure can be
216 written as Eq. (9).

$$E_{sint} = \begin{bmatrix} 1 & -R_{sint} \\ 0 & 1 \end{bmatrix} \text{ and } E_{sext} = \begin{bmatrix} 1 & -R_{sext} \\ 0 & 1 \end{bmatrix} \quad (9)$$

217

218 The internal and external surface resistances of the building envelope can be computed by Eqs. (10) and
219 (11), respectively.

220

$$\text{Internal surface resistance, } R_{sint} = \frac{1}{(1.2F_e h_r + h_c)} \quad (10)$$

221

$$\text{External surface resistance, } R_{sext} = \frac{1}{h_c + F_e h_r} \quad (11)$$

222

223 The homogeneous wall enclosure transmission matrix with internal and external surface film resistances
 224 can be represented by Eq. (12).

$$\begin{bmatrix} T_{int} \\ q_{int} \end{bmatrix} = \begin{bmatrix} 1 & -R_{sint} \\ 0 & 1 \end{bmatrix} \begin{bmatrix} f_1 & f_2 \\ f_3 & f_1 \end{bmatrix} \begin{bmatrix} 1 & -R_{sext} \\ 0 & 1 \end{bmatrix} \begin{bmatrix} T_{ext} \\ q_{ext} \end{bmatrix} \quad (12)$$

225 The composite wall enclosure transmission matrix can be written as in Eq. (13),

$$\begin{bmatrix} T_{int} \\ q_{int} \end{bmatrix} = \begin{bmatrix} 1 & -R_{sint} \\ 0 & 1 \end{bmatrix} \begin{bmatrix} f_1 & f_2 \\ f_3 & f_1 \end{bmatrix} \begin{bmatrix} g_1 & g_2 \\ g_3 & g_1 \end{bmatrix} \dots \begin{bmatrix} 1 & -R_{sext} \\ 0 & 1 \end{bmatrix} \begin{bmatrix} T_{ext} \\ q_{ext} \end{bmatrix} \quad (13)$$

226 The walled enclosure layers are represented by f and g.

227 The building enclosure transmission matrix can be represented, as shown in Eqs. (14) and (15).

$$\begin{bmatrix} T_{int} \\ q_{int} \end{bmatrix} = \begin{bmatrix} E_{11} & E_{12} \\ E_{21} & E_{22} \end{bmatrix} \begin{bmatrix} T_{ext} \\ q_{ext} \end{bmatrix} \quad (14)$$

228

$$\begin{bmatrix} E_{11} & E_{12} \\ E_{21} & E_{22} \end{bmatrix} = \begin{bmatrix} 1 & -R_{sint} \\ 0 & 1 \end{bmatrix} \begin{bmatrix} A_1 + jA_2 & (A_3 + jA_4)/a \\ (-A_4 + jA_3).a & A_1 + jA_2 \end{bmatrix}_f \begin{bmatrix} A_1 + jA_2 & (A_3 + jA_4)/a \\ (-A_4 + jA_3).a & A_1 + jA_2 \end{bmatrix}_g \dots \begin{bmatrix} 1 \\ 0 \end{bmatrix} \quad (15)$$

229

230 where E_{11} , E_{12} , E_{21} , and E_{22} are the matrix components, which are complex numbers.

231 2.3.2 Steady Thermal transmittance (U)

232 The thermal transmittance of the wall enclosure is the reciprocal of its thermal resistance. It is
 233 the ratio of heat transfer through a building envelope to the difference in temperature across the
 234 envelope. A lower the thermal transmittance value of the wall envelope results in higher thermal
 235 insulation of the envelope as a steady-state quantity. It is represented in Eq. (17) as

236

$$U = \frac{1}{R_{sext} + \left(\frac{x_1}{k_1}\right) + \left(\frac{x_2}{k_2}\right) + R_{air} \dots + \dots R_{sint}} \quad (16)$$

237 where X_1 and X_2 are the wall layer thicknesses, and k_1 and k_2 are the thermal conductivity values of
 238 layer-1 and layer-2, respectively.

239 **2.3.3 Thermal admittance (Y)**

240 The envelope material's capability to absorb heat from the atmosphere and release it back to the
241 same over some time is known as thermal admittance. This indicates the thermal mass of the enclosure.
242 They can be computed by Eq. (18).

$$Y = \left| \left(\frac{q_{int}}{T_{int}} \right)_{T_{ext}=0} \right| = |-E_{11}/E_{12}| \quad (17)$$

243

244 **2.3.4 Attenuation factor (μ) and its decrement delay (ϕ)**

245 The attenuation factor is the difference between the outside and inside wall enclosure temperature
246 swings. The attenuation factor is inversely proportional to the difference between outside and inside
247 temperature swings. Lower attenuation factor or decrement factor results in higher thermal heat capacity
248 or thermal mass. Decrement lag or Time lag or phase shift is the delay in the heat flow from the outside
249 to the inside of the wall enclosure measured in hours. For thin structures with low thermal capacity
250 values, the value of the attenuation factor is unity, and the value of decrement delay is zero. The
251 attenuation factor decreases while decrement delay increases with increasing thermal capacity for
252 structural materials (Alavez-Ramirez et al., 2014). These values can be determined according to Eq. (19)
253 and (20)

$$\mu = \left| -\frac{1}{UE_{12}} \right| \quad (18)$$

$$\phi = \frac{12}{\pi} \arctan \left(\frac{\text{Im} \left(-\frac{1}{UE_{12}} \right)}{\text{Re} \left(-\frac{1}{UE_{12}} \right)} \right) \quad (19)$$

254

255

256 **2.3.1 Unsteady transmittance (u_{cyc})**

257 The unsteady transmittance of the wall enclosure is the amount of fluctuating heat gain in the wall
258 envelope. The low value of unsteady transmittance implies the low fluctuating heat gain in the wall
259 envelope. It can be computed by the following Eq. (20):

$$u_{cyc} = \left| -\frac{1}{E_{12}} \right| \quad (20)$$

260 Thermal characteristics of the envelopes (i.e., walls and roofs) are unsteady. Above Eq. (16) to Eq. (20)
261 were used to find dynamic thermal characteristics of the wall envelope (Davies, 2004).

262
263 A MATLAB program was developed to compute the dynamic thermal quantities of homogeneous and
264 composite wall enclosures exposed to various atmospheric wind velocities. Method validation was
265 carried out for both homogeneous and composite wall envelopes against published CIBSE results. The
266 deviation of the calculated results from the CIBSE values was observed to be less than $\pm 1\%$ for
267 homogeneous envelopes and less than $\pm 2.5\%$ for composite envelopes. Computation of the dynamic
268 thermal quantities of wall enclosures by the admittance method requires thermophysical properties of
269 the wall materials (thermal conductivity, specific heat, and density), the thickness of the wall materials
270 (x), the outside wall surface heat transfer coefficient (h_{sext}), and the inside wall surface heat transfer
271 coefficient (h_{sint}). The thicknesses of the composite walls (multilayer walls) considered in this paper are
272 presented in Fig. 2. The outside wall surface heat transfer coefficient (h_{sext}) depends on the atmospheric
273 wind velocity, while the inside wall surface coefficient (h_{sint}) was considered at standstill wind velocities
274 as per CIBSE standards. The external wall envelopes were exposed to atmospheric wind velocities of
275 0.0, 0.5, 0.7, 2, 4, 6, 8, and 10 m/s. Table 2 shows the influence of wind velocity on external and internal
276 surface heat transfer coefficients. The above parameters were used as input for the admittance method to
277 obtain dynamic thermal quantities such as admittance, attenuation factor, and decrement delay.

278 **Table 2** Influence of wind velocity on external and internal surface heat transfer coefficients of
279 wall enclosures.
280

281 In this paper, we present the dynamic thermal performance of seven composite wall configurations
282 (multilayer walls) of burnt brick, mud brick, laterite stone, and cinder concrete with expanded
283 polystyrene insulation material exposed to eight atmospheric wind speeds ($n=28$ (7
284 (configurations/designs) \times 4 (building materials) = total of 28 combinations of composite walls)). The
285 arrangement of the multilayers is given in Fig. 2. The optimum design of composite walls under severe
286 wind velocity conditions is proposed for reduced cooling costs in buildings.

287 **2.4 Energy Economic Analysis:**

288 **2.4.1 Heating degree-hours and cooling degree-hours**

289 Heating degree-hours and cooling degree-hours are used to calculate the energy required for heating and
290 cooling of the building. The base temperature is the point at which the mechanical system should be
291 switched on to provide thermal comfort inside the environment. As per the ASHRAE standard, the
292 minimum base temperature is considered to be 23.3 °C for cooling and heating of the built environment.
293 The energy needed for cooling or heating depends upon the difference between the outside temperature
294 and base temperature. The sol air temperature depends upon absorptivity and solar radiation on the
295 building surface, which is considered based on the outside temperature. Cooling degree-hours or days
296 are calculated by the summing the hours or days when the outside sol air temperature is above the base
297 temperature. Similarly, heating degree-hours or days are the sum of the difference between the outside
298 sol air temperature and the base temperature over a particular period of hours or days. ASHRAE
299 metrological data have been used for cooling and heating degree-hours in Chennai climatic conditions
300 (13.0827°N, 80.2707°E). Chennai is located in a hot and humid climatic zone of India. The annual

301 degree days and hours of Chennai has been presented in Fig. 3. Chennai has only cooling degree
302 days/cooling degree hours, and this climate does not have heating degree days or heating degree hours.
303 Therefore, air-conditioning costs associated with building cooling are significant in this climatic zone.

304
305 **Fig. 3.** Monthly heating and cooling degree days and hours of Chennai (13.0827⁰N, 80.2707⁰E).

306
307 The total number of cooling degree-days or hours (CDD or CDH) and heating degree-days or hours
308 (HDD or HDH) can be calculated using Eqs. (21-24)

$$CDD = \sum_{l=1}^{N_{CD}} (T_{SO} - T_{ba}) = N_{CD} \cdot \Delta T, \text{ for } T_{so} \geq T_{ba} \quad (21)$$

309

$$CDH = \sum_{l=1}^{N_{CH}} (T_{SO} - T_{ba}) = N_{CH} \cdot \Delta T, \text{ for } T_{so} \geq T_{ba} \quad (22)$$

310

$$HDD = \sum_{l=1}^{N_{HD}} (T_{SO} - T_{ba}) = N_{HD} \cdot \Delta T, \text{ for } T_{so} \geq T_{ba} \quad (23)$$

311

$$HDH = \sum_{l=1}^{N_{HH}} (T_{SO} - T_{ba}) = N_{HH} \cdot \Delta T, \text{ for } T_{so} \geq T_{ba} \quad (24)$$

312 Where, N_{CD} , N_{CH} , N_{HD} , and N_{HH} are the number of cooling days, number of cooling hours, number of
313 heating days, and number of heating hours, respectively. T_{so} is the sol air temperature, and it depends on
314 the outside air temperature, surface absorption, and solar radiation on the surface.

315 **2.4.2 Building material cost and annual energy cost savings**

316 Table 3 shows the building material costs of various building materials with their dimensions.
317 The mud brick is the cheapest building material among all four building materials studied. The thermo-
318 economic method was proposed by Duffie et al. (1985). Annual energy-cost savings is the yearly
319 cooling cost savings obtained when using insulation. It can be calculated using Eq. (25). Table 4 shows
320 the parameters used for the energy economic analysis to obtain annual energy cost savings, life cycle
321 saving costs, and payback periods.

$$ESC = \frac{10^{-3} E.CDH.\Delta u_{cyc}}{COP} \quad (25)$$

322 Where Δu_{cyc} is the difference in unsteady thermal transmittance with and without insulation.

323 **Table 3** Building material cost of various building materials.

324 **Table 4** Parameters used for energy-economic analysis.

325

326

327 2.4.3 Life cycle cost savings

328 Life cycle cost savings is the total energy savings throughout the lifetime calculated by including the
329 insulation cost and building material cost. It can be calculated using Eq. (26).

$$LCS = \frac{10^{-3}.P_1.E.CDH.\Delta u_{cyc}}{COP} - C_{inv} \quad (26)$$

330 Present worth factor (P_1) depends upon the inflation rate (f), discount rate (D), and life cycle period of
331 building materials (N). It can be calculated using Eq. (27). Investment cost (C_{inv}) is the sum of insulation
332 cost (C_{ins}) and building material cost (C_{bm}).

$$P_1 = \frac{1}{D-f} \left[1 - \left(\frac{1+f}{1+D} \right)^N \right] \quad (27)$$

333 **2.4.4 Payback period**

334 Payback period refers to the time (usually in years) that it takes to recover the initial investment of
335 insulation cost and building material cost. It can be calculated using Eq. (28)

$$PB = \frac{C_{inv}}{ESC} \quad (28)$$

336

337 **3. Results and Discussion**

338 **3.1. Unsteady transmittance and admittance of various wall designs and wall materials**

339 The steady transmittance (U) is not accurate measure of thermal performance as it takes only thermal
340 conductivity property in calculations under steady state conditions as in Eq. (16). In practice, the
341 properties (Specific heat capacity and density) contributing to thermal mass should also be considered
342 under periodic heat transfer conditions for the computation of accurate unsteady thermal transmittance
343 (u_{cyc}) and air-conditioning cost savings as presented in Eqs. (20) & (25). The findings of both higher
344 admittance values and lower unsteady transmittance values imply the reduced heat fluctuation gain in
345 the building. Fig. 4 (a) shows the unsteady transmittance and admittance of composite burnt brick wall
346 enclosures exposed to various external heat transfer coefficients. The configurations that save the most
347 energy from a higher admittance (lower unsteady transmittance) perspective across all seven studied
348 configurations at all external surface heat transfer coefficients and wind velocities are the burnt brick
349 composite wall envelope with an expanded polystyrene insulation layer positioned at the center of the
350 burnt brick envelope (C-H₃) and the burnt brick composite wall envelope with one part of the expanded
351 polystyrene insulation layer positioned at the external side, and another positioned at the center (C-H₆).
352 At 10 m/s external wind velocity, the burnt brick composite wall envelope with an expanded

353 polystyrene insulation layer positioned at the center of the burnt brick envelope (C-H₃) has an
354 admittance value of 4.898 W/m²K and an unsteady transmittance value of 0.39W/m²K. At 10 m/s
355 external wind velocity, the burnt brick composite wall enclosure with one part of the expanded
356 polystyrene insulation layer positioned at the external side and another positioned at the center (C-H₆)
357 had an admittance value of 4.779 W/m²K and an unsteady transmittance value of 0.21W/m²K. The burnt
358 brick composite wall envelope without an insulation layer (C-H₁) had admittance and unsteady
359 transmittance values of 4.5744 W/m²K and 1.21 W/m²K, respectively. For any building wall, high
360 thermal admittance indicates high thermal mass and low unsteady thermal transmittance, leading to high
361 thermal storage. Among all seven studied configurations, C-H₃ and C-H₆ are preferable at all external
362 surface heat transfer coefficients/wind velocities due to the high admittance values at low unsteady
363 transmittance.

364
365 Fig. 4 (b) presents the unsteady transmittance and admittance of composite mudbrick wall enclosures
366 exposed to various external heat transfer coefficients. C-H₃ and C-H₆ are the most energy-saving
367 configurations at all external surface heat transfer coefficients and wind velocities from a higher
368 admittance (lower unsteady transmittance) perspective among the seven studied configurations. At 10
369 m/s external wind velocity, C-H₆ has a 4.6798 W/m²K admittance value and a 0.29 W/m²K unsteady
370 transmittance value. In contrast, C-H₁ has a 4.4832 W/m²K admittance value and a 1.26 W/m²K
371 unsteady transmittance value.

372
373 Fig. 4 (c) shows the unsteady transmittance and admittance of composite laterite wall enclosures at all
374 external surface heat transfer coefficients and wind velocities. C-H₃ and C-H₆ configurations provide the
375 most energy savings from a higher admittance (lower unsteady transmittance) perspective among the
376 seven studied configurations at all external wind velocities. At 10 m/s external wind velocity, C-H₃ has

377 an admittance value of 5.3902 W/m²K and a 0.41 W/m²K unsteady transmittance value, while C-H₆ has
378 a 5.2664 W/m²K admittance value and an unsteady transmittance value of 0.20 W/m²K. C-H₁ has
379 admittance and unsteady transmittance values of 5.0247 W/m²K and 1.64 W/m²K, respectively.

380
381 Fig. 4 (d) presents the unsteady transmittance and admittance of composite cinder concrete wall
382 enclosures. C-H₃ and C-H₆ configurations provide most energy savings from a higher admittance (lower
383 unsteady transmittance) perspective for all the external surface heat transfer coefficients and wind
384 velocities among the seven studied configurations. At 10 m/s external wind velocity, C-H₃ has an
385 admittance value of 4.4929 W/m²K and an unsteady transmittance value of 0.39 W/m²K, while C-H₆ has
386 admittance and unsteady transmittance values of 4.3812 W/m²K and 0.21 W/m²K, respectively. C-H₁
387 has an admittance value of 4.1905 W/m²K and an unsteady transmittance value of 1.15 W/m²K.

388 For any building wall, higher thermal admittance indicates a higher thermal mass, while lower
389 unsteady thermal transmittance indicates higher thermal insulation. Among the seven configurations, C-
390 H₃ and C-H₆ are preferable for all external surface heat transfer coefficients and wind speeds due to the
391 higher admittance values at lower unsteady transmittance.

392
393 **Fig. 4.** Unsteady transmittance and admittance of composite wall enclosures for various external surface
394 heat transfer coefficients: (a) burnt brick, (b) mudbrick, (c) laterite stone, and (d) cinder concrete.

395
396

397 **3.2 Attenuation factor and decrement delay of various wall designs and wall materials**

398 Fig. 5(a) shows the attenuation factor of composite burnt brick wall enclosures at various external
399 surface heat transfer coefficients. C-H₅ and C-H₆ provide the most energy savings from a lower
400 attenuation factor point of view at all external wind velocities and across all seven studied

401 configurations. At 10 m/s external wind velocity, C-H₅ has the lowest attenuation factor of 0.1896, and
402 C-H₆ has a low attenuation factor of 0.2041. C-H₁ has the highest attenuation factor value of 0.5172.
403 Fig. 5(b) shows the decrement delay of composite burnt brick wall enclosures at various external surface
404 heat transfer coefficients. C-H₅ and C-H₇ provide the most energy savings with a higher decrement delay
405 at all external wind velocities across all seven studied configurations. At a 10 m/s external wind
406 velocity, C-H₅ has the highest decrement delay of 9.9059 h, and C-H₇ also has a high decrement delay of
407 8.9944 h. C-H₁ has the lowest decrement delay value of 6.4079 h.

408
409 **Fig. 5.** (a) Attenuation factor, and (b) decrement delay of composite burnt brick wall enclosures at
410 various external surface heat transfer coefficients.

411 Fig. 6(a) presents the attenuation factor of composite mudbrick wall enclosures for various external
412 surface heat transfer coefficients. C-H₅ and C-H₆ provide the most energy-savings from a lower
413 attenuation factor point of view across all seven configurations at all external wind velocities. At 10 m/s
414 external wind velocity, C-H₅ has the lowest attenuation factor of 0.1965, and C-H₆ also has a low
415 attenuation factor of 0.2147. C-H₁ has the highest attenuation factor value of 0.5219.

416 Fig. 6(b) presents the decrement delay of composite mudbrick wall enclosures at various external
417 surface heat transfer coefficients. C-H₅ and C-H₇ configurations provided the most energy savings from
418 a higher decrement lag perspective across the seven configurations and at all external wind velocities. At
419 10 m/s external wind velocity, C-H₅ has the highest decrement lag of 9.8365 h, and C-H₇ also has a high
420 decrement lag of 8.955 h. C-H₁ has the lowest decrement delay value of 6.4245 h.

421
422

423 **Fig. 6.** (a) Attenuation factor, and (b) decrement delay of composite mudbrick wall enclosures at various
424 external surface heat transfer coefficients.

425

426 Fig. 7(a) shows the attenuation factor of composite laterite wall enclosures at various external surface
427 heat transfer coefficients. C-H₅ and C-H₆ provided the most energy savings from a lower attenuation
428 factor point of view across all seven studied configurations and at all external wind velocities. At 10 m/s
429 external wind velocity, C-H₅ has the lowest attenuation factor of 0.1737 due to a higher thermal mass
430 offered by the insulation position. C-H₆ also has a low attenuation factor of 0.1738. C-H₁ has the highest
431 attenuation factor value of 0.5364 because of the lower thermal mass of the wall.

432 Fig. 7(b) shows the decrement delay of composite laterite wall enclosures at various external surface
433 heat transfer coefficients. C-H₅ and C-H₇ provide the most energy savings from a higher decrement lag
434 perspective across the seven studied configurations and at all external wind velocities. At 10 m/s
435 external wind velocity, C-H₅ has the highest decrement lag of 9.7889 h, and C-H₇ also has a high
436 decrement lag of 8.7059 h. C-H₁ has the lowest decrement lag value of 5.8129 h.

437 **Fig. 7.** (a) Attenuation factor, and (b) Decrement delay of composite laterite wall enclosures at various
438 external surface heat transfer coefficients.

440 Fig. 8 (a) shows the attenuation factor of composite cinder concrete walls at various external surface
441 heat transfer coefficients. C-H₅ and C-H₆ provide the most energy savings with the lowest attenuation
442 factor among the seven studied configurations at all external wind velocities. At 10 m/s external wind
443 velocity, C-H₅ has the lowest attenuation factor of 0.2544, and C-H₆ also has a low attenuation factor of
444 0.2941. C-H₁ has the highest attenuation factor value of 0.6044.

445 Fig. 8(b) shows the decrement delay of composite cinder concrete walls at various external surface
446 heat transfer coefficients. C-H₅ and C-H₇ provide the most energy savings with a higher decrement lag
447 across all seven studied configurations at all external wind velocities. At 10 m/s external wind velocity,
448 C-H₅ has the highest decrement lag of 8.857 h, and C-H₇ also has a high decrement lag of 8.0459 h. C-
449 H₁ has the lowest decrement delay value of 5.6926 h.

450

451

452 **Fig. 8.** (a) Attenuation factor, and (b) decrement delay of composite cinder concrete wall enclosures at
453 various external surface heat transfer coefficients.

454

455 **3.3. Annual energy cost savings and life cycle cost savings of various wall designs and wall** 456 **materials**

457 Fig. 9 shows that the life cycle cost savings dependent upon the position of the insulation and quantity
458 of insulation material. Because the quantity of insulation material is constant for the designs of C-H₂ to
459 C-H₇, the cost of insulation should be maintained consistently in all those configurations. The burnt
460 brick with expanded polystyrene and the resulting six configurations (e.g., from C-H₂ to C-H₇) show
461 annual energy cost savings of 1.18, 1.03, 1.03, 1.25, 1.22, and 1.15 \$/m², respectively. Among all
462 configurations studied, C-H₅ shows the highest annual energy cost savings for the four different
463 materials (i.e., burnt brick, mud brick, laterite stone, and cinder concrete with expanded polystyrene) of
464 1.25, 1.20, 1.71, and 1.29 \$/m², respectively. The C-H₅ configuration also shows the highest life cycle
465 cost savings for the burnt brick, mud brick, laterite stone, and cinder concrete expanded polystyrene as
466 10.95, 11.16, 18.32, and 11.62 \$/m², respectively. Among all the configuration studied, C-H₅ is found as
467 the best performer for all building materials.

468

469

470 **Fig. 9.** Impact of configuration on insulation cost and life cycle cost savings.

471 Fig. 10 shows that the increase in the wind velocity or external heat transfer coefficient for the
472 configuration (C-H₅) leads to an increase in annual energy cost savings and life cycle cost savings. As
473 the wind velocity increased from 0 to 10m/s, there is increase in the annual energy cost savings of burnt
474 brick, mud brick, laterite stone and cinder concrete with expanded polystyrene in the order 103%

475 (increase from 0.72 to 1.46 $\$/m^2$), 98% (increase from 0.70 to 1.38 $\$/m^2$), 133% (increase from 0.9 to
476 2.09 $\$/m^2$) and 84% (increase from 0.81 to 1.47 $\$/m^2$), respectively. At 2 m/s wind speed, the burnt
477 brick, mud brick, laterite stone, and cinder concrete recorded the annual energy cost savings of 1.25,
478 1.20, 1.71, and 1.29 $\$/m^2$, respectively. As the wind velocity increased from 0 to 10 m/s, there is
479 increase in the life cycle cost savings of burnt brick, mud brick, laterite stone, and cinder concrete with
480 expanded polystyrene in the order of 345% (from 3.16 to 14.11 $\$/m^2$), 264% (from 3.79 to 13.80 $\$/m^2$),
481 277% (from 6.35 to 23.97 $\$/m^2$) and 218% (from 4.49 to 14.28 $\$/m^2$), respectively. The increase in the
482 annual energy cost savings and life cycle cost savings is significant in the wind speed range 0 to 2 m/s.
483 In contrast, the increase in the values of annual energy cost savings and life cycle cost savings is gradual
484 in the wind speed range 2 m/s to 10 m/s. At 2 m/s wind speed, the burnt brick, mud brick, laterite stone,
485 and cinder concrete have life cycle cost savings of 10.95, 11.16, 18.32, and 11.62 $\$/m^2$, respectively.
486 Among all building materials studied, the laterite with expanded polystyrene and configuration (C-H₅)
487 showed the highest life cycle cost savings and the highest annual energy cost savings at every wind
488 speed.

489
490 **Fig. 10.** Annual energy cost savings and life cycle cost savings of various wall designs.

491 Fig. 11 shows building, insulation, energy saving, and life cycle saving costs of CH-5 wall
492 enclosure at 2 m/s external wind velocity. The recommended order for the highest energy cost savings is
493 laterite, cinder concrete, burnt brick, and mud brick. The life cycle cost saving considers both energy
494 cost savings and investment costs of building materials. The recommended order for the highest life
495 cycle saving costs is laterite stone, cinder concrete, mud brick, and burnt brick. The mud brick shows
496 better life cycle costs than burnt brick due to its lowest investment cost.

497
498 **Fig. 11.** Building, insulation, energy saving, and life cycle saving costs of CH-5 wall enclosure at 2 m/s
499 external wind velocity.

500 3.4 Payback periods of various wall designs and wall materials

501 Fig. 12 shows that the increase in wind velocity leads to a reduction in the payback period. As
502 the wind velocity increased from 0 to 10 m/s, the reductions in the payback period of burnt brick, mud
503 brick, laterite, and cinder concrete with expanded polystyrene are 50.63% (reduction from 10.37 to 5.12
504 years), 49.0% (reduction from 9.2 to 4.77 years), 57.1% (reduction from 7.67 to 3.3 year), and 45.11%
505 (reduction from 9.4 to 5.04 years), respectively. At 2 m/s wind speed, the burnt brick, mud brick, laterite
506 stone, and cinder concrete have payback periods of 6, 5.48, 4.03, and 5.74 years, respectively. Among
507 all building materials studied, the laterite stone with expanded polystyrene has the lowest payback
508 period at all wind speeds.

509

510 **Fig. 12.** Payback periods of building materials.

511

512 4. Conclusions

513 In this paper, we considered seven different types of cost-effective external wall designs relevant to
514 the manufacture of dwellings in emerging economies. Specifically, we studied the effect of surface
515 external heat transfer coefficients on thermal performance characteristics of the walls and their air-
516 conditioning cost-savings potential. Four different types of building materials with expanded
517 polystyrene insulation were considered with seven different outer enclosure configurations. The main
518 conclusions of the study are:

- 519 • C-H₅ (with one insulation layer at the external side and another at the internal) is the best wall
520 envelope design configuration because it provides a lower attenuation factor, as well as highest
521 decrement lag values at all wind velocities among the seven, studied wall enclosure designs. Hence,
522 this wall enclosure design is the best to reduce heat gain by convection.

523 • The variable with the greatest effect on outer building enclosure performance is the material used.
524 The laterite with C-H₅ enclosure design offers the highest annual energy cost savings (1.71 \$/m² at
525 2 m/s), highest life cycle cost savings (18.32 \$/m² at 2m/s) and the lowest payback period (4.03 year
526 at 2 m/s) at all wind speeds as compared to the other studied building materials. The preference
527 order of building materials for high net annual cost savings and low payback periods is found as:
528 laterite stone > cinder concrete > mud brick > burnt brick.

529 • A study of the thermal performance parameters is essential for non-air-conditioned buildings,
530 whereas the study of annual energy cost savings, life cycle costs, and payback periods are essential
531 for air-conditioned buildings. The preference order of the wall designs for improved thermal
532 performance parameters in non-air-conditioned buildings is C-H₅ > C-H₆ > C-H₂ > C-H₇ > C-H₄ >
533 C-H₃ > C-H₁, and this order applies very well to air-conditioned buildings as well.

534 • The most significant changes in the thermal performance characteristics, life cycle cost savings,
535 annual energy cost savings, and payback periods occur between wind speeds of 0 and 2 m/s. Above
536 2 m/s wind speed, the changes in thermal performance characteristics and energy economic
537 parameters are gradual.

538 The results of the work will help inform the development of energy-conscious yet cost-effective
539 buildings in emerging economies.

540

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542

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Nomenclature			
C_{bm}	Building material cost ($\$/m^2$)	P	Time period [s]
C_{ins}	Insulation cost ($\$/m^2$)	P_1	Present worth factor
C_{inv}	Investment cost ($\$/m^2$)	q_{ext}	External heat flux of complex values at $x=0$ [W/m^2]
C_p	Specific heat [J/kgK]	q_{int}	Internal heat flux of complex values at $x=X$ [W/m^2]
C_s	Wind speed [m/s]	R_{air}	Air resistance [m^2K/W]
D	Discount rate [%]	R_{sint}	Internal surface resistance [m^2K/W]
E	Electricity price ($\$/kWh$)	R_{sext}	External surface resistance [m^2K/W]
$E_{11}, E_{12}, E_{21}, E_{22}$	Elements of composite wall transmission matrix	t	Time [s]
f	Inflation rate (%)	T_b	Base temperature [$^{\circ}C$]
f_1, f_2, f_3, f_4	Elements of homogeneous wall matrix	T_{ext}	External temperature variation [$^{\circ}C$]
Fe	Emissivity factor[-]	T_{int}	Internal temperature variation [$^{\circ}C$]
g_1, g_2, g_3, g_4	Elements of homogeneous wall matrix	T_{so}	sol air temperature [$^{\circ}C$]
hc	Convective heat transfer coefficient [$W/m^2 K$]	u_{cyc}	Cyclic transmittance [W/m^2K]
hr	Radiative heat transfer coefficient [$W/m^2 K$]	U	Thermal transmittance [W/m^2K]
h_{sext}	External surface heat transfer coefficient [$W/m^2 K$]	R_t	Thermal resistance with insulation [m^2K/W]
h_{sint}	Internal surface heat transfer coefficient [$W/m^2 K$]	X	Thickness of the wall [m]
k	Thermal Conductivity [W/mK]	x	Finite thickness [m]
K	Geometry constant	Y	Admittance [W/m^2K]
l	Insulation thickness (m)		
Greek symbols			

α	Thermal diffusivity [m^2/s]	ρ	density [kg/m^3]
ε	Emissivity of the wall [-]	σ	Boltzmann constant [$5.67 \times 10^{-8} \text{ W}/\text{m}^2 \text{ K}^4$]
μ	Attenuation factor [-]	ϕ	Decrement lag [h]
Abbreviations			
BB	Burnt brick	HDD	Heating degree days
CNC	Cinder concrete	HDH	Heating degree-hours
COP	Coefficient of performance	LS	Laterite stone
CDD	Cooling degree days	LSC	Life cycle saving cost
CDH	cooling degree-hours	MB	Mud-brick
EP	Expanded polystyrene	P	Cement plaster
ESC	Annual energy saving cost	PB	Payback period

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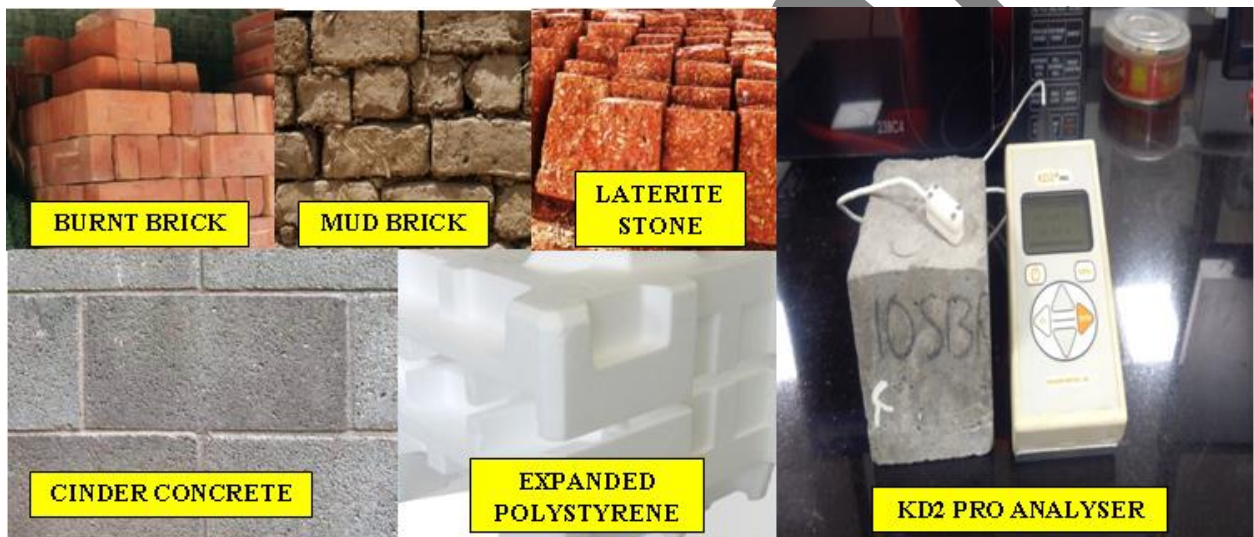
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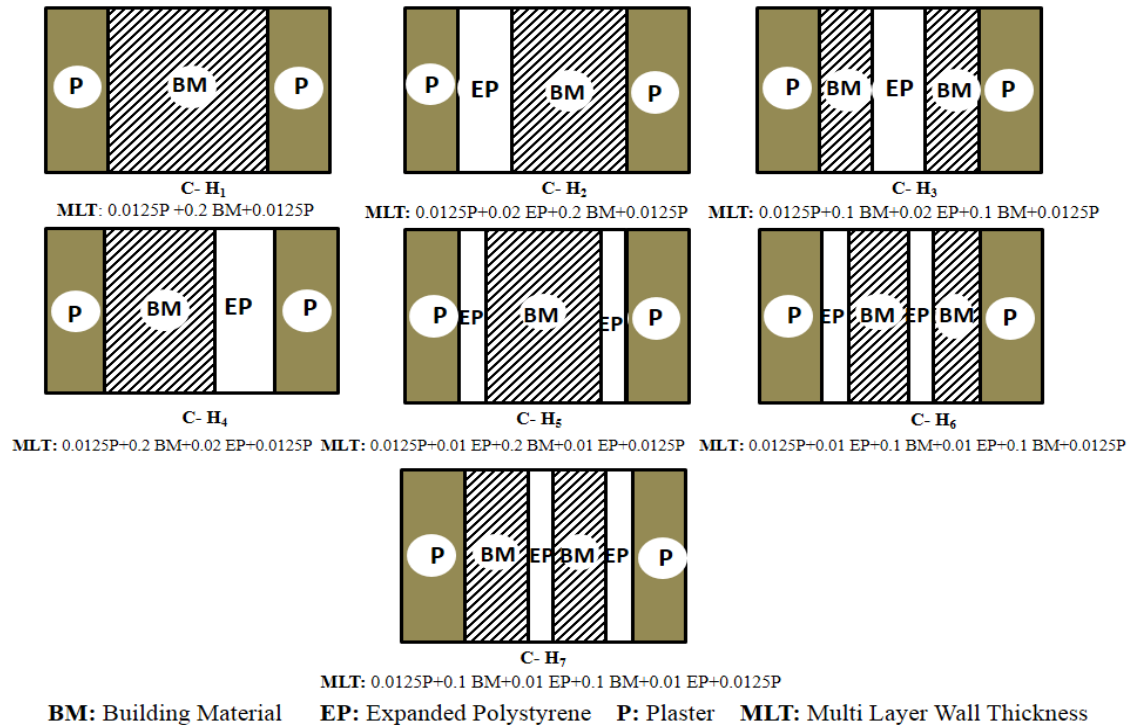
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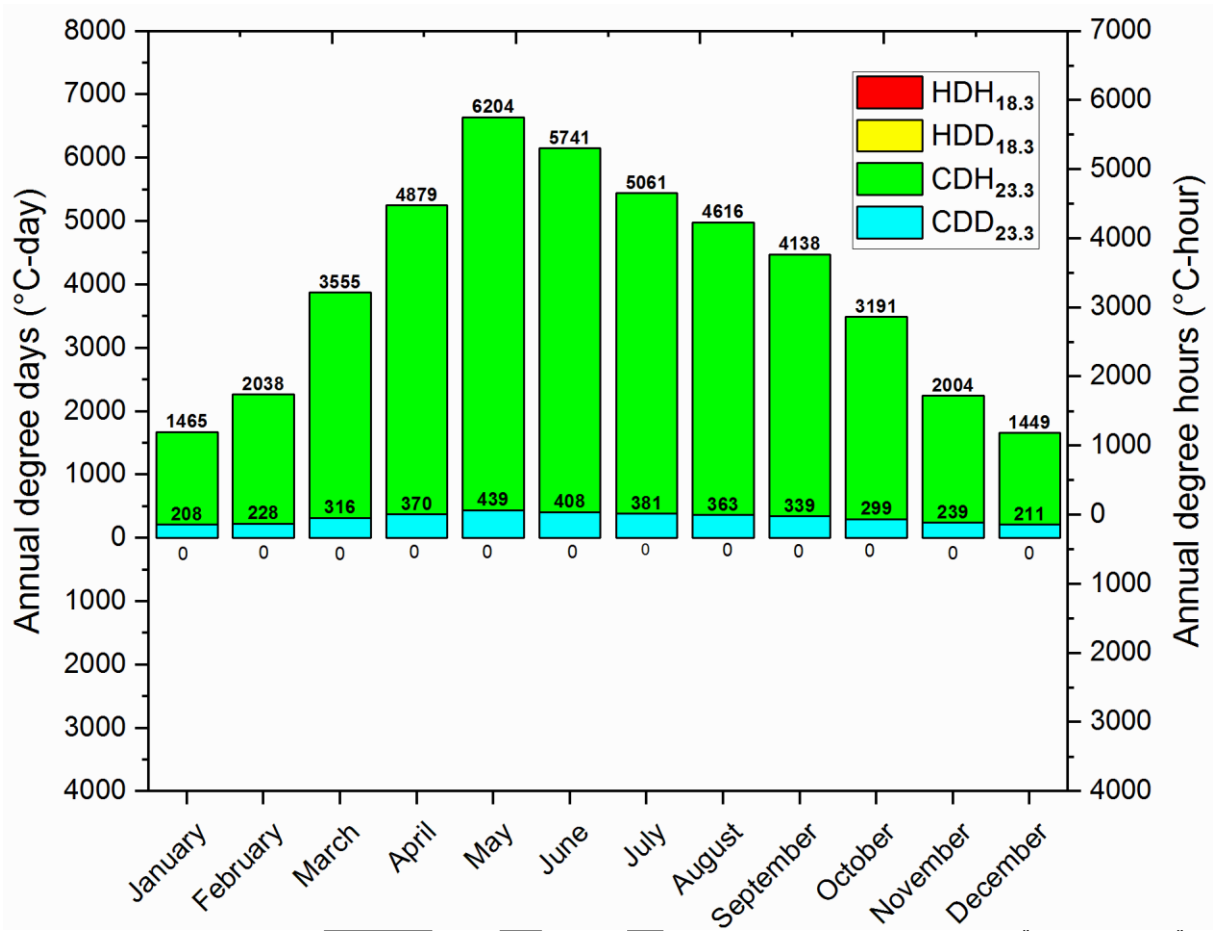
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Fig. 1. Building and insulation materials of wall enclosures.

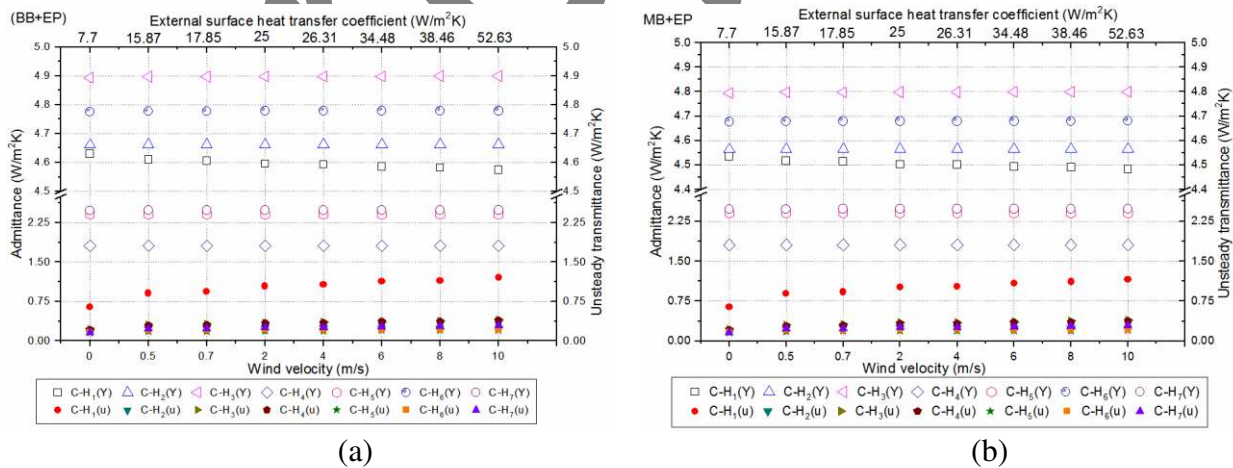


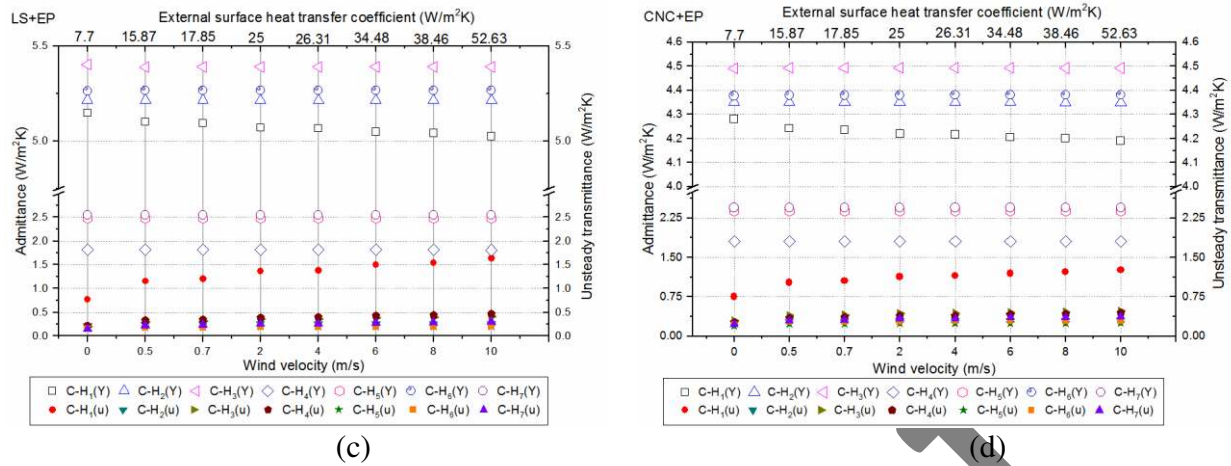
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Fig. 2. Configuration of composite wall enclosures and expanded polystyrene insulation considered in this study.



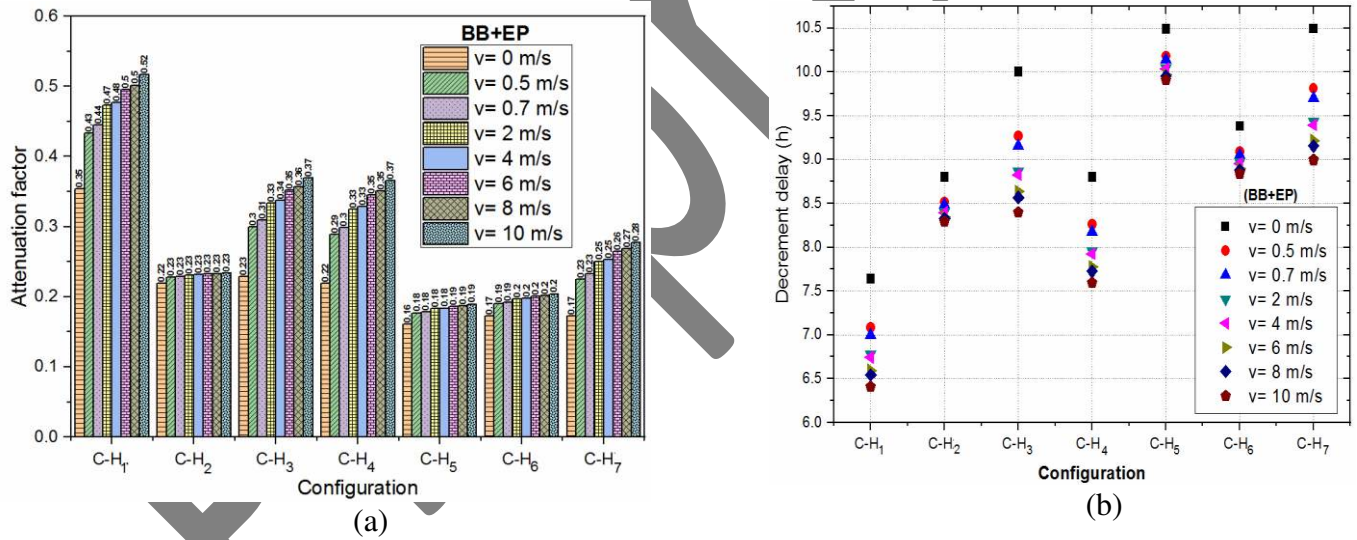
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676 **Fig. 3.** Monthly heating and cooling degree days and hours of Chennai (13.0827⁰N, 80.2707⁰E).
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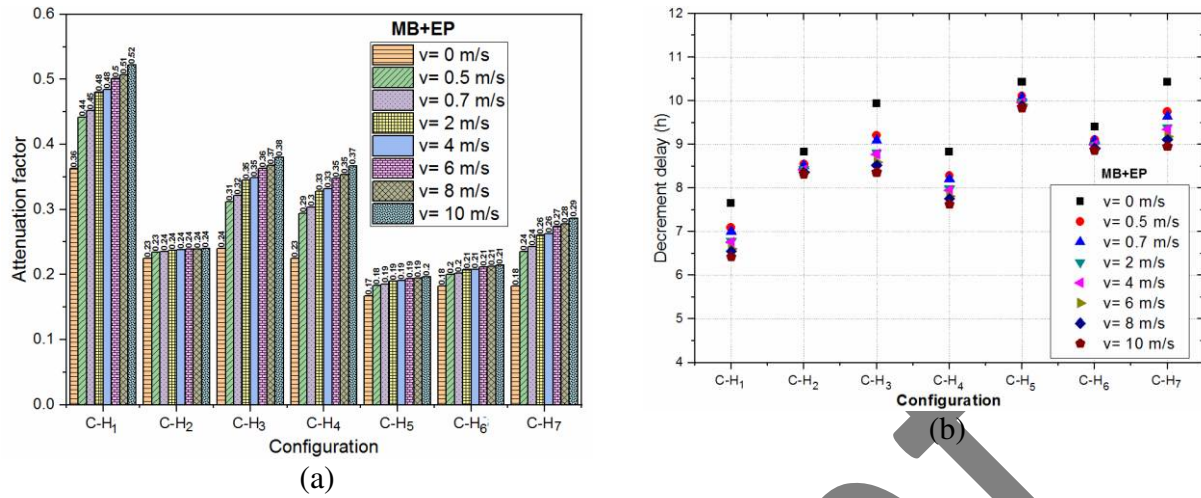


678 **Fig. 4.** Unsteady transmittance and admittance of composite wall enclosures for various external surface
 679 heat transfer coefficients: (a) burnt brick, (b) mudbrick, (c) laterite stone, and (d) cinder concrete.

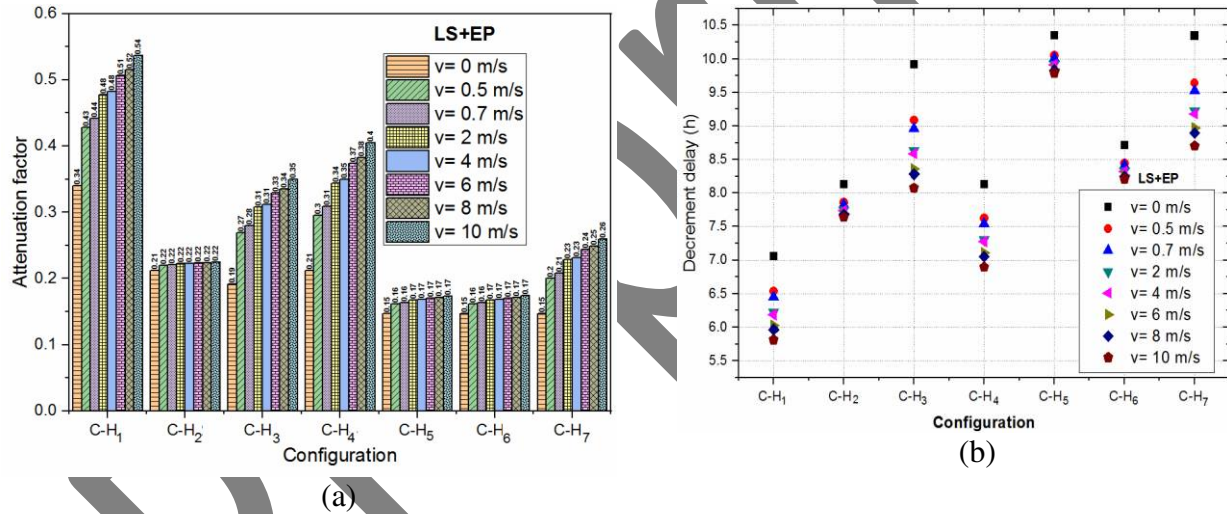
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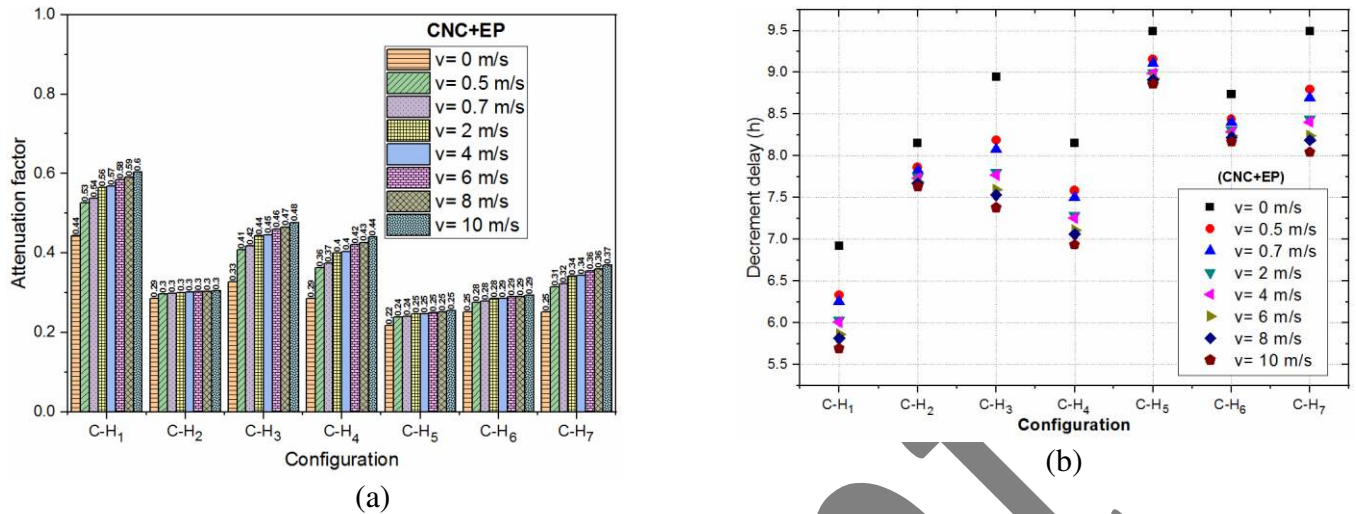
681 **Fig. 5.** (a) Attenuation factor, and (b) decrement delay of composite burnt brick wall enclosures at
 682 various external surface heat transfer coefficients.
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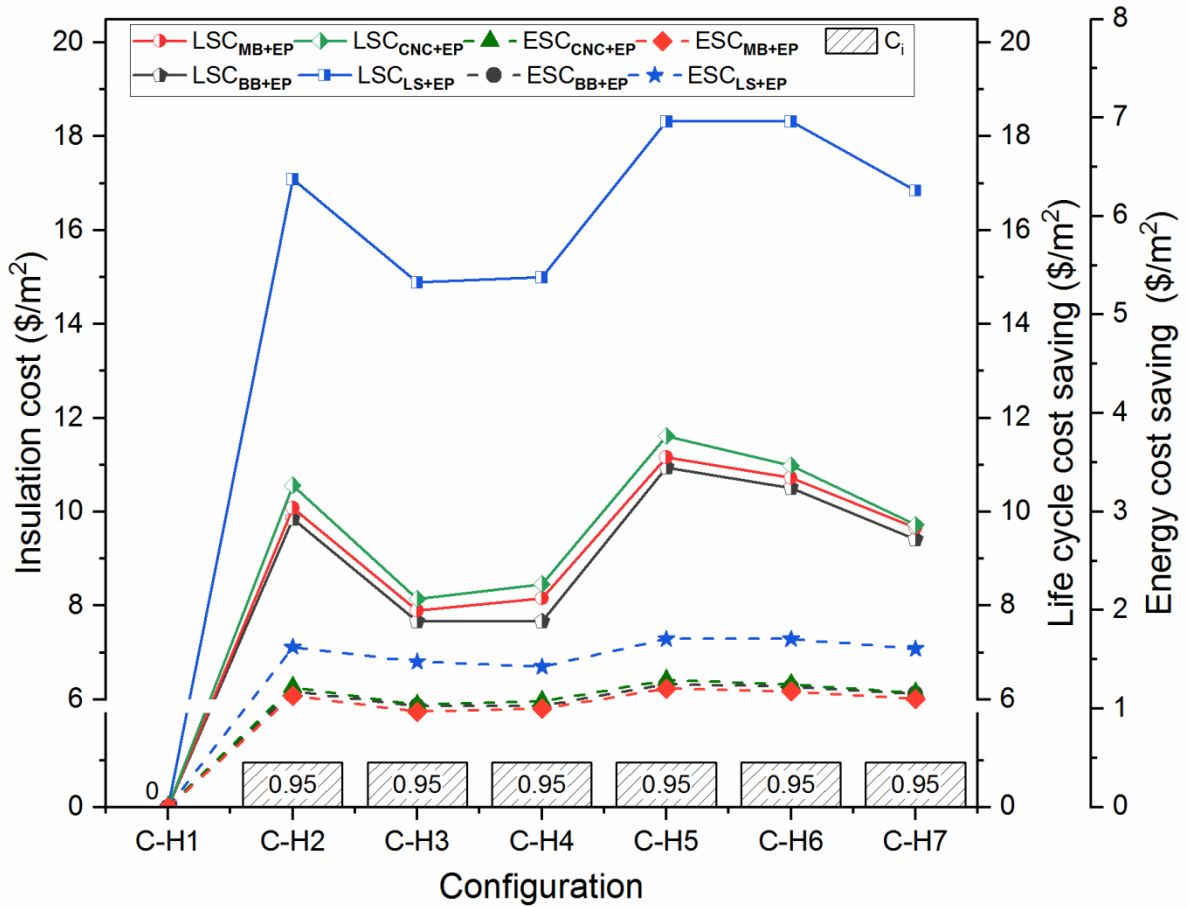
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 685 **Fig. 6.** (a) Attenuation factor, and (b) decrement delay of composite mudbrick wall enclosures at various
 686 external surface heat transfer coefficients.



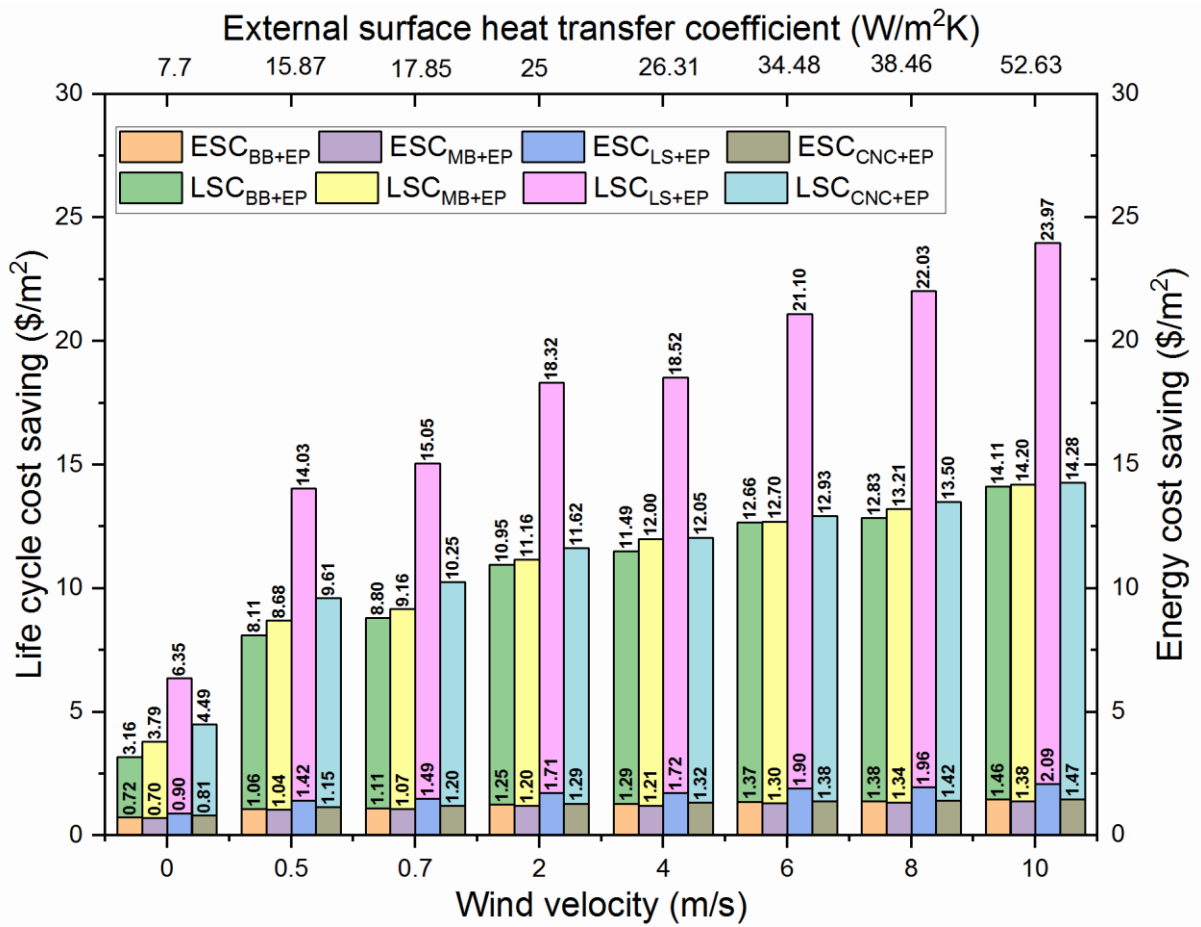
687 **Fig. 7.** (a) Attenuation factor, and (b) Decrement delay of composite laterite wall enclosures at various
 688 external surface heat transfer coefficients.



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691 **Fig. 8.** (a) Attenuation factor, and (b) decrement delay of composite cinder concrete wall enclosures at
692 various external surface heat transfer coefficients.



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694 **Fig. 9.** Impact of configuration on insulation cost and life cycle cost savings.

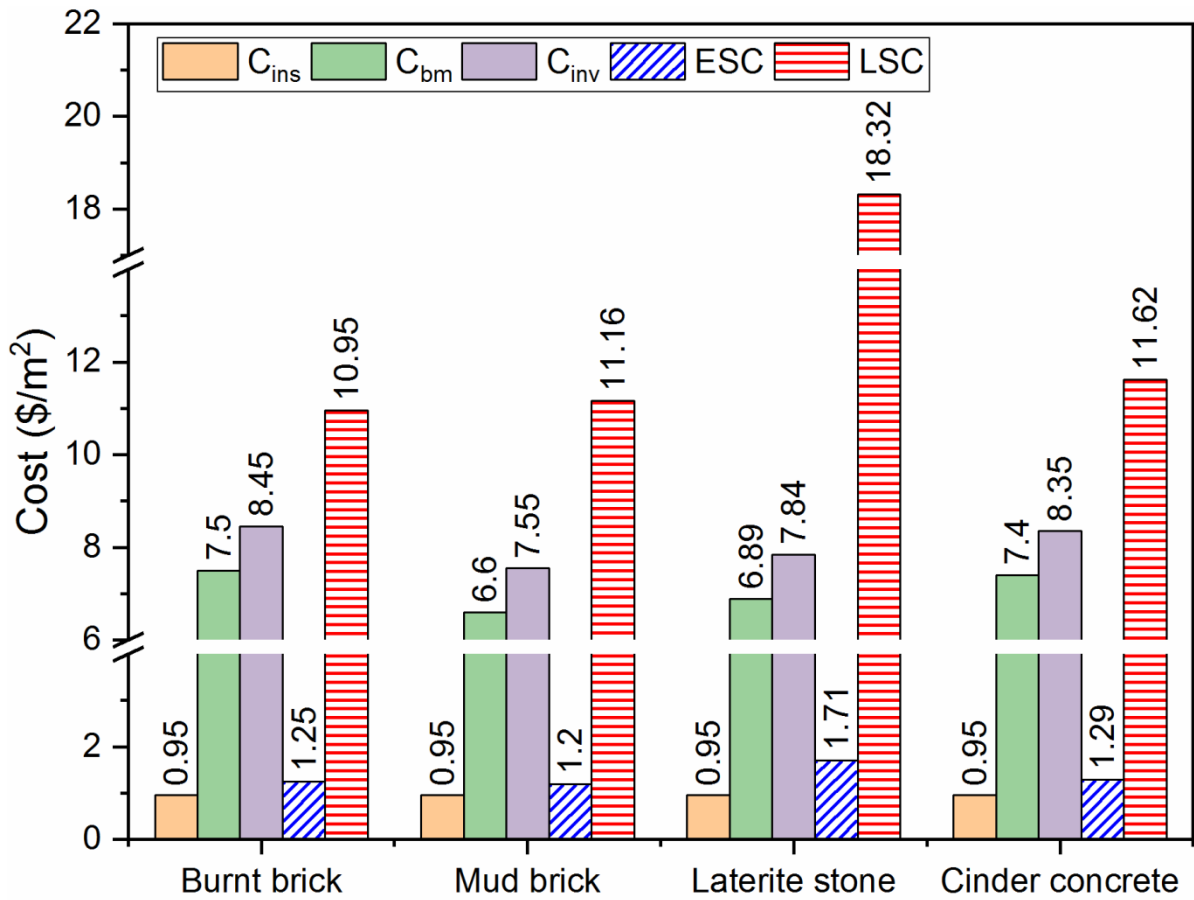


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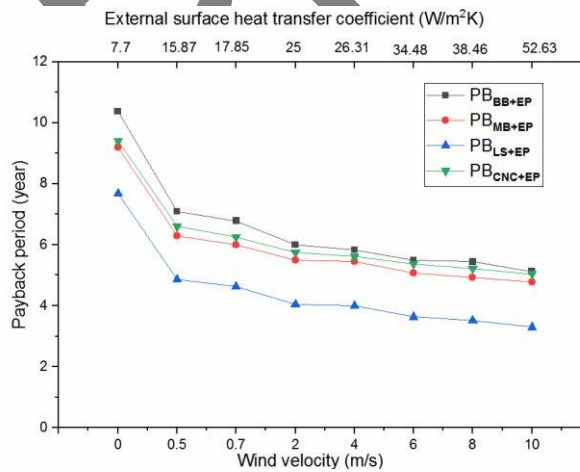
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Fig. 10. Annual energy cost savings and life cycle cost savings of various wall designs.

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697
 698 **Fig. 11.** Building, insulation, energy, and life cycle saving costs of CH-5 wall enclosure at 2 m/s
 699 external wind velocity.



700
 701 **Fig. 12.** Payback periods of building materials.
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703

704 **List of tables**

705 **Table 1** Thermophysical properties of building wall enclosure materials.

S. No.	Building material	Code	Thermal conductivity k [W/mK]	Density ρ [kg/m ³]	Specific heat C _p [J/kgK]
1.	Burnt brick	BB	0.811±0.003	1820±7	880±0.02
2.	Mudbrick	MB	0.75±0.002	1731±6	880±0.01
3.	Laterite stone	LS	1.369±0.004	1000±4	1926±0.04
4.	Cinder concrete	CNC	0.686±0.003	1406±3	840±0.03
5.	Expanded polystyrene	EP	0.038±0.001	16±1	1340±0.05
6.	Cement plaster	P	0.721±0.002	1762±2	840±0.02

706

707 **Table 2** Influence of wind velocity on external and internal surface heat transfer coefficients of
708 wall enclosures.

S. No.	Atmospheric wind velocity at the outer side of the wall enclosure [m/s]	External surface heat transfer coefficient $h_{sext} = \frac{1}{R_{sext}}$ [W/m ² K]
1.	Standstill	7.70
2.	0.5	15.87
3.	0.7	17.85
4.	2	25.00
5.	4	26.31
6.	6	34.48
7.	8	38.46
8.	10	52.63

709

710 **Table 3** Building material cost of various building materials

S. No.	Building materials	Dimensions (m) (LBH)	No. of brick required /m ²	\$/m ²
1.	Burnt brick	0.23 x 0.101 x 0.076	108	7.5
2.	Mud brick	0.23 x 0.101 x 0.076	108	6.6
3.	Laterite stone	0.305 x 0.2 x 0.2	15	6.89

4.	Cinder concrete	0.41 x 0.2 x 0.2	11	7.4
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Table 4 Parameters used for energy-economic analysis.

S. No.	Parameter	Value
1.	Cooling degree-hours (CDH)	44343 °C- h
2.	Heating degree-hours (HDH)	0
3.	Unit cost of electricity (E)	0.082 \$/kWh (Kumar et al., 2018)
4.	Discount rate (D)	6.25%
5.	Inflation rate (f)	3.44%
6.	Insulation thickness (<i>l</i>)	0.02 m
7.	Insulation cost (<i>C_{ins}</i>)	0.95 \$/m ²
8.	Coefficient of performance (COP)	2.5 (Kumar et al., 2018)
9.	Life cycle period of building materials (N)	20 (Yu et al., 2009)
10.	Present worth factor (<i>P_f</i>)	14.77

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